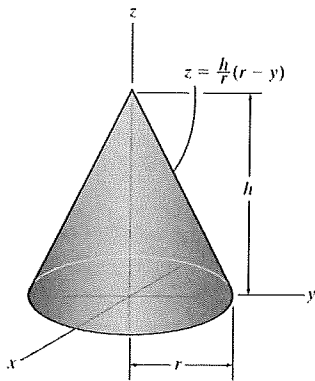


ME 230 Kinematics and Dynamics

HW # 7 Solution

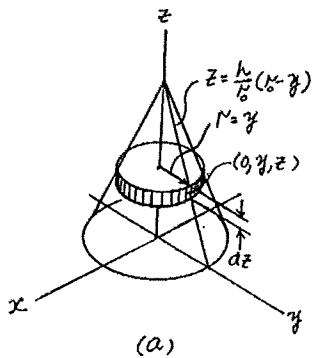
17-6



The mass of the disk element shown in Fig. is $dm = \rho dV = \rho \pi r^2 dz$. Here $r = y = r_0 - \frac{r_0}{h} z$. Thus, $dm = \rho \pi \left(r_0 - \frac{r_0}{h} z \right)^2 dz$.

The mass moment of inertia of this element about the z axis is

$$dI_z = \frac{1}{2} dm r^2 = \frac{1}{2} (\rho \pi r^2 dz) r^2 = \frac{1}{2} \rho \pi r^4 dz = \frac{1}{2} \rho \pi \left(r_0 - \frac{r_0}{h} z \right)^4 dz$$



The mass of the cone:

$$M = \int dm = \int_0^h \rho \pi \left(r_0 - \frac{r_0}{h} z \right)^2 dz = \rho \pi \left[\frac{1}{3} \left(r_0 - \frac{r_0}{h} z \right)^3 \left(-\frac{h}{r_0} \right) \right] \Big|_0^h = \frac{1}{3} \rho \pi r_0^2 h$$

$$\text{Integrating } dI_z, I_z = \int dI_z = \int_0^h \frac{1}{2} \rho \pi \left(r_0 - \frac{r_0}{h} z \right)^4 dz = \frac{1}{2} \rho \pi \left[\frac{1}{5} \left(r_0 - \frac{r_0}{h} z \right)^5 \left(-\frac{h}{r_0} \right) \right] \Big|_0^h = \frac{1}{10} \rho \pi r_0^4 h$$

$$M = \frac{1}{3} \rho \pi r_0^2 h, I_z = \frac{1}{10} (\rho \pi r_0^2 h) r_0^2 = \frac{3}{10} M r_0^2$$

17-23

$$M_c = 7.85(10^3)(0.05\pi(0.01)^2) = 0.1233 \text{ kg}$$

$$M_p = 7.85(10^3)(0.03 \times 0.180) \times 0.02 = 0.8478 \text{ kg}$$

$$\begin{aligned} I_x &= \left[\frac{1}{2}(0.1233 \times 0.01^2) \right] + \left[\frac{1}{2}(0.1233)(0.01)^2 \right. \\ &\quad \left. + (0.1233)(0.12)^2 \right] + \left[\frac{1}{12}(0.8478)(0.03)^2 + (0.180)^2 \right. \\ &\quad \left. + (0.8478)(0.06)^2 \right] \\ &= 0.00719 \text{ kg} \cdot \text{m}^2 = 7.19 \text{ g} \cdot \text{m}^2 \end{aligned}$$

17-27

$$\Sigma F_x = m(a_G)_x; \quad 20 = \left(\frac{600}{32.2} \right) a_G$$

$$a_G = 1.0733 \text{ ft/s}^2 = 1.07 \text{ ft/s}^2$$

$$\begin{aligned} \Sigma + \Sigma M_A &= \Sigma (M_K)_A; \quad 20(4) - 600(0.5) + 2N_B(15) \\ &= \frac{600}{32.2} (1.0733) \times 2 \end{aligned}$$

$$N_B = 86.7 \text{ lb}$$

$$+\uparrow \Sigma F_y = m(a_G)_y; \quad 2N_A + 2(86.7) - 600 = 0$$

$$N_A = 213 \text{ lb}$$

17-33

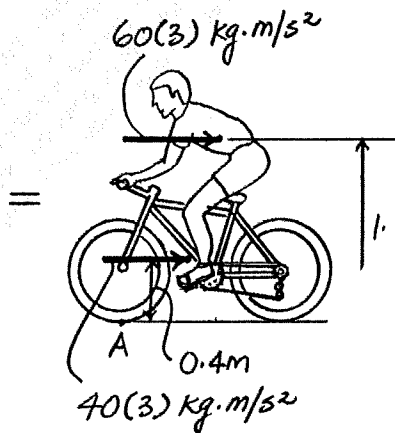
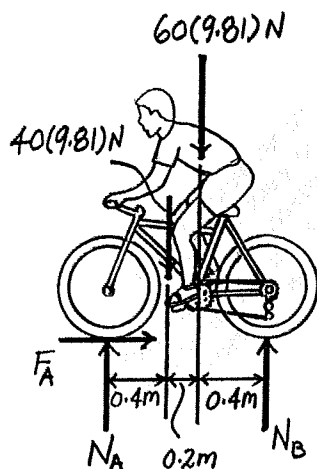
$$+\Sigma M_A = (M_K)_A;$$

$$N_B(1) - 40(9.81)(0.4) - 60(9.81)(0.6) \\ = -60(3)(1.25) - 40(3)(0.4)$$

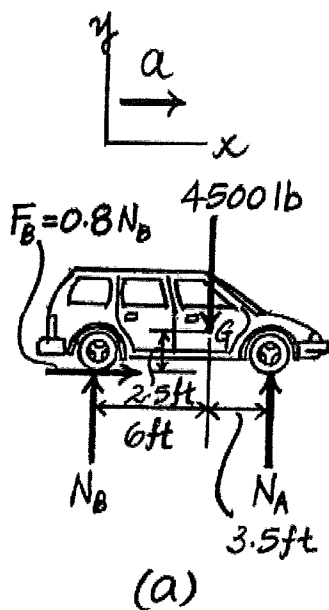
$$N_B = 237.12 \text{ N}$$

$$+\uparrow \Sigma F_y = m(a_g)_y; N_A + 237.12 - 40(9.81) \\ - 60(9.81) = 0$$

$$N_A = 743.88 \text{ N}$$



17-38



$$① \pm \Sigma F_x = m(a_g)_x; 0.8 N_B = \left(\frac{4500}{32.2}\right) a_{\max}$$

$$② +\uparrow \Sigma F_y = m(a_g)_y; N_A + N_B - 4500 = 0$$

$$③ +\Sigma M_G = 0; N_A(3.5) + 0.8 N_B(2.5) \\ - N_B(6) = 0$$

Solving eq. ①, ②, ③,

$$\text{Ans: } N_A = 2.4 \text{ Kip, } N_B = 2.10 \text{ Kip,}$$

$$a_{\max} = 12.02 \frac{\text{ft}}{\text{s}^2} = 12.0 \frac{\text{ft}}{\text{s}^2}$$

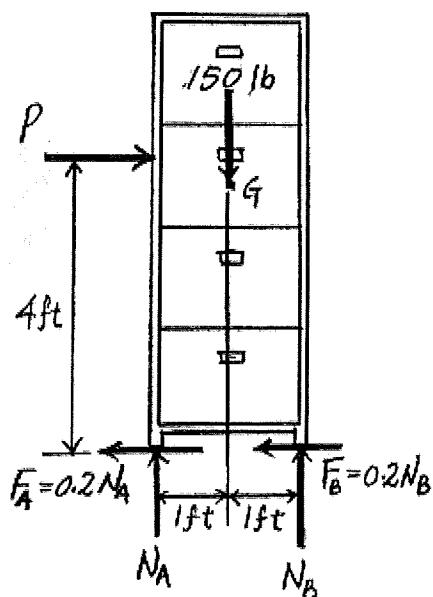
17-43

P: unknown minimum force needed.

Assume the cabinet slides before it tips.

$$F_A = \mu_s N_A = 0.2 N_A$$

$$F_B = \mu_s N_B = 0.2 N_B$$



(a)

$$\pm \sum F_x = 0; \quad P - 0.2 N_A - 0.2 N_B = 0 \quad (1)$$

$$+\uparrow \sum F_y = 0; \quad N_A + N_B - 150 = 0 \quad (2)$$

$$+\sum M_A = 0; \quad N_B (2) - 150 (1) - P (4) = 0 \quad (3)$$

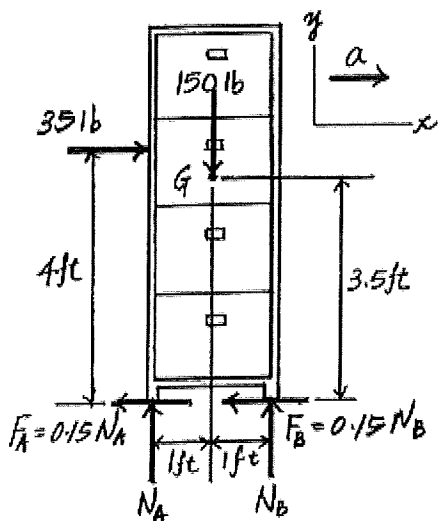
Solving Eq. (1), (2), (3),

$$P = 30 \text{ lb}, \quad N_A = 15 \text{ lb}, \quad N_B = 135 \text{ lb}$$

Since $P < 35 \text{ lb}$, N_A is positive $\Rightarrow$ the cabinet will slide.

$$F_A = \mu_k N_A = 0.15 N_A, \quad F_B = \mu_k N_B = 0.15 N_B$$

Referring to the FBD,



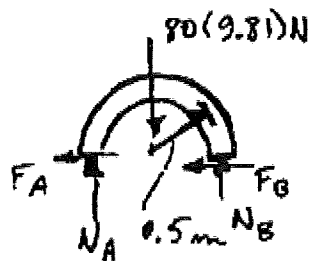
$$\pm \sum F_x = m(a_g)_x; \quad 35 - 0.15 N_A - 0.15 N_B = \left(\frac{150}{32.2}\right)a$$

$$\pm \sum F_y = m(a_g)_y; \quad N_A + N_B = 150$$

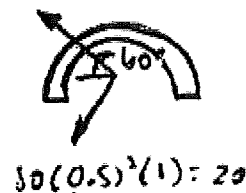
$$+\sum M_G = 0; \quad N_B (1) - 0.15 N_B (3.5) - 0.15 N_A (3.5) - N_A (1) - 35 (0.5) = 0,$$

$$a = 2.68 \text{ ft/s}^2, \quad N_A = 26.9 \text{ lb}, \quad N_B = 123 \text{ lb}$$

17-53



$$80(0.25)(1) = 20$$



$$+\uparrow \Sigma F_y = m(a_g)_y;$$

$$N_A + N_B - 80(9.81) = 20 \sin 60^\circ - 20 \cos 60^\circ$$

$$N_A + N_B = 792.12$$

$$(+\Sigma M_A = \Sigma (M_k)_A;$$

$$N_B(1) - 80(9.81)(0.5) = 20 \cos 60^\circ(0.2) - 20 \cos 60^\circ(0.5) + 20 \sin 60^\circ(0.2)$$

$$N_B = 402 \text{ N}, \quad N_A = 391 \text{ N} \quad \#$$

17-74

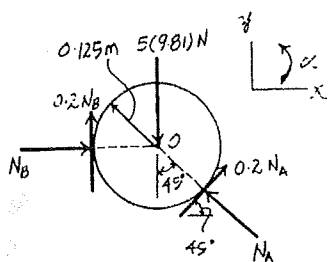
(see 17-59 next page)

$$\text{mass moment of inertia } I_o = \frac{1}{2} m r^2 = \frac{1}{2} \times 5 \times 0.125^2 = 0.0390625 \text{ kg} \cdot \text{m}^2$$

$$(1) \pm \Sigma F_x = m(a_g)_x; \quad N_B + 0.2 N_A \cos 45^\circ - N_A \sin 45^\circ = 0$$

$$(2) +\uparrow \Sigma F_y = m(a_g)_y; \quad 0.2 N_B + 0.2 N_A \sin 45^\circ + N_A \cos 45^\circ - 5(9.81) = 0$$

$$(3) (+\Sigma M_o = I_o \alpha; \quad 0.2 N_A \times 0.125 - 0.2 N_B (0.125) = 0.0390625 \alpha$$



Solving Eq (1), (2), (3),

$$N_A = 51.01 \text{ N}, \quad N_B = 28.85 \text{ N}$$

$$\alpha = 14.2 \text{ rad/s}^2 \quad \#$$

17-59

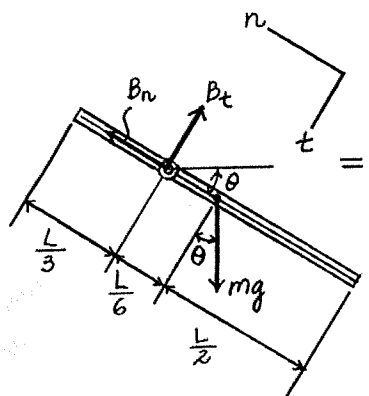
The rod rotate about a fixed axis

$$(a_G)_t = \alpha r_G = \alpha \left(\frac{L}{6}\right), (a_G)_n = \omega^2 r_G = \omega^2 \left(\frac{L}{6}\right)$$

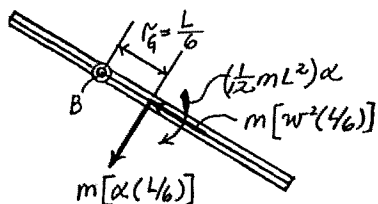
The mass moment of inertia of the rod about G

$$I_G = \frac{1}{12} mL^2$$

$$+\Sigma M_B = \Sigma (M_k)_B; -mg \cos \theta \left(\frac{L}{6}\right) = -m \left[\alpha \left(\frac{L}{6}\right) \right] \left(\frac{L}{6}\right) - \left(\frac{1}{12} mL^2\right) \alpha, \quad \alpha = \frac{3g \cos \theta}{2L}$$



$$\left(\text{or } \Sigma M_B = I_B \alpha, \text{ where } I_B = \frac{1}{12} mL^2 + m \left(\frac{L}{6}\right)^2 = \frac{1}{9} mL^2 \right. \\ \left. + \Sigma M_B = I_B \alpha; -mg \cos \theta \left(\frac{L}{6}\right) = -\left(\frac{1}{9} mL^2\right) \alpha \right. \\ \left. \alpha = \frac{3g \cos \theta}{2L} \right)$$



The force equation of motion along n and t axes:

$$\Sigma F_t = m(a_G)_t; mg \cos \theta - B_t = m \left[\left(\frac{3g \cos \theta}{2L} \right) \left(\frac{L}{6} \right) \right]$$

$$B_t = \frac{3}{4} mg \cos \theta$$

$$\Sigma F_n = m(a_G)_n; B_n - mg \sin \theta = m \left[\omega^2 \left(\frac{L}{6} \right) \right]$$

$$B_n = \frac{1}{6} m \omega^2 L + mg \sin \theta$$

$$\text{Kinematics: } \int \omega d\omega = \int \alpha d\theta, \int_0^\omega \omega d\omega = \int_0^\theta \frac{3g \cos \theta}{2L} d\theta, \omega = \sqrt{\frac{3g}{L} \sin \theta}$$

$$\text{When } \theta = 90^\circ, \omega = \sqrt{\frac{3g}{L}}; B_t = \frac{3}{4} mg \cos 90^\circ = 0$$

$$B_n = \frac{1}{6} m \left(\frac{3g}{L} \right) L + mg \sin 90^\circ = \frac{3}{2} mg$$

$$F_A = \sqrt{A_t^2 + A_n^2} = \sqrt{0^2 + \left(\frac{3}{2} mg \right)^2} = \frac{3}{2} mg \quad \#$$

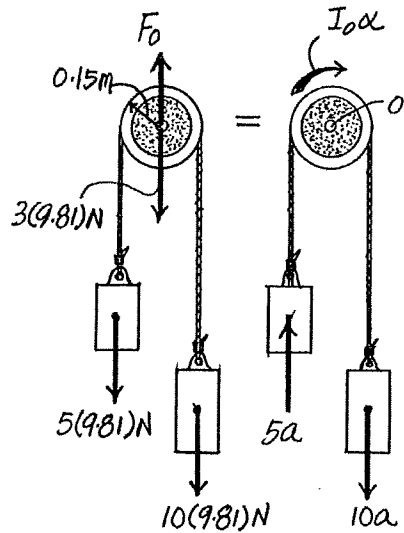
17-79

Angular acceleration of the pulley:

$$\alpha = \frac{a}{r} = \frac{a}{0.15} = 6.667a$$

Mass moment of inertia of the pulley:

$$I_0 = \frac{1}{2} M r^2 = \frac{1}{2} (3) (0.15)^2 = 0.03375 \text{ kg} \cdot \text{m}^2$$



$$\begin{aligned} \sum M_0 &= \sum (M_k)_0; \quad 5(9.81)(0.15) - 10(9.81)(0.15) \\ &= -0.03375(6.667a) - 5a(0.15) \\ &\quad - 10a(0.15) \end{aligned}$$

$$a = 2.973 \text{ m/s}^2 = 2.97 \frac{\text{m}}{\text{s}^2}$$

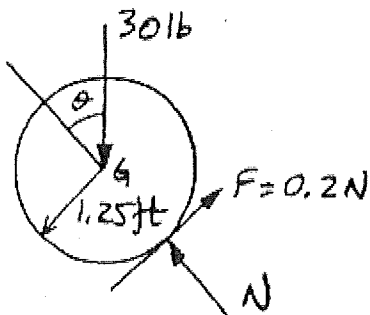
17.95

$$\textcircled{1} + \sum F_x = m(a_G)_x; \quad 30 \sin \theta - 0.2 \text{ N} = \left(\frac{30}{32.2} \right) (1.25 \alpha)$$

$$\textcircled{2} + \sum F_y = m(a_G)_y; \quad N - 30 \cos \theta = 0$$

$$\textcircled{3} \sum M_C = I_G \alpha; \quad 0.2 \text{ N} (1.25) = \left[\frac{30}{32.2} (0.6)^2 \right] \alpha$$

Solving $\textcircled{1}, \textcircled{2}, \textcircled{3}$,



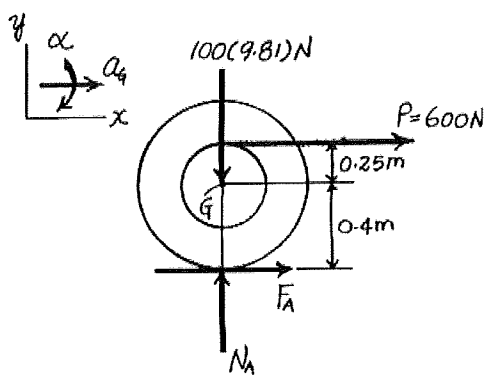
$$30 \sin \theta - 6 \cos \theta = 26.042 \cos \theta$$

$$30 \sin \theta = 32.042 \cos \theta$$

$$\tan \theta = 1.068$$

$$\theta = 46.9^\circ$$

17-98



$$\pm \Sigma F_x = m(a_G)_x; 600 + F_A = 100 a_G$$

$$+\uparrow \Sigma F_y = m(a_G)_y; N_A - 100(9.81) = 0$$

$$+\Sigma M_G = I_G \alpha; 600(0.25) - F_A(0.4) = [100(0.3)^2] \alpha$$

Assuming no slipping; $a_G = 0.4 \alpha$

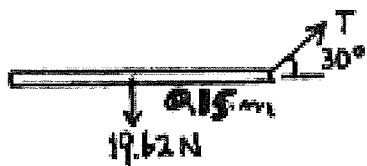
$$\alpha = 15.6 \text{ rad/s}^2 \#$$

$$a_G = 6.24 \text{ m/s}^2, N_A = 981 \text{ N}, F_A = 24 \text{ N}$$

$$\text{Since } (F_A)_{\max} = 0.2(981) = 196.2 \text{ N} > 24 \text{ N}$$

OK

17-102



$$\pm \Sigma F_x = m(a_G)_x; T \cos 30^\circ = 2(a_G)_x$$

$$+\uparrow \Sigma F_y = m(a_G)_y; T \sin 30^\circ - 19.62 = 2(a_G)_y$$

$$+\Sigma M_G = I_G \alpha; T \sin 30^\circ (0.15) = \left[\frac{1}{12} (2) (0.3)^2 \right] \alpha$$

$$\bar{a}_B = \bar{a}_G + \bar{a}_{B/G}$$

$$a_B \sin 30^\circ \bar{i} - a_B \cos 30^\circ \bar{j} = (a_G)_x \bar{i} + (a_G)_y \bar{j} + \alpha (0.15) \bar{j}$$

$$(\rightarrow) (a_B) \sin 30^\circ = (a_G)_x$$

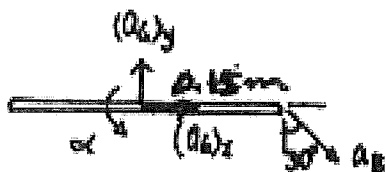
$$(+\uparrow) (a_B) \cos 30^\circ = -(a_G)_y - \alpha (0.15)$$

$$1.7321(a_G)_x = -(a_G)_y - 0.15 \alpha$$

$$T = 5.61 \text{ N} \#$$

$$(a_G)_x = 2.43 \text{ m/s}^2, (a_G)_y = -8.41 \text{ m/s}^2$$

$$\alpha = 28.0 \text{ rad/s}^2 \#$$



17-109

mass moment of inertia:

$$I_G = mr^2 = 500(0.5^2) = 125 \text{ kg} \cdot \text{m}^2$$

$$\sum M_A = \sum (M_k)_A; 0 = 125\alpha - 500 a_G(0.5)$$

$$\bar{a}_G = \bar{a}_A + \alpha \times \bar{r}_{G/A} - \omega^2 \bar{r}_{G/A}$$

$$\bar{a}_G \bar{i} = 3\bar{i} + (a_A)_n \bar{j} + (\alpha \bar{k} \times 0.5\bar{j}) - \omega^2(0.5\bar{j})$$

$$a_G \bar{i} = (3 - 0.5\alpha)\bar{i} + [(a_A)_n - 0.5\omega^2]\bar{j}$$

$$\begin{cases} a_G = 3 - 0.5\alpha \\ 0 = 125\alpha - 500 a_G \end{cases} \Rightarrow \begin{cases} a_G = 1.5 \text{ m/s}^2 \rightarrow \\ \alpha = 3 \text{ rad/s}^2 \end{cases}$$

