

# Lecture #1

Smits 5.1 & 5.2

user name: me431/599  
password: fluid

want to solve equations to get the flow field.

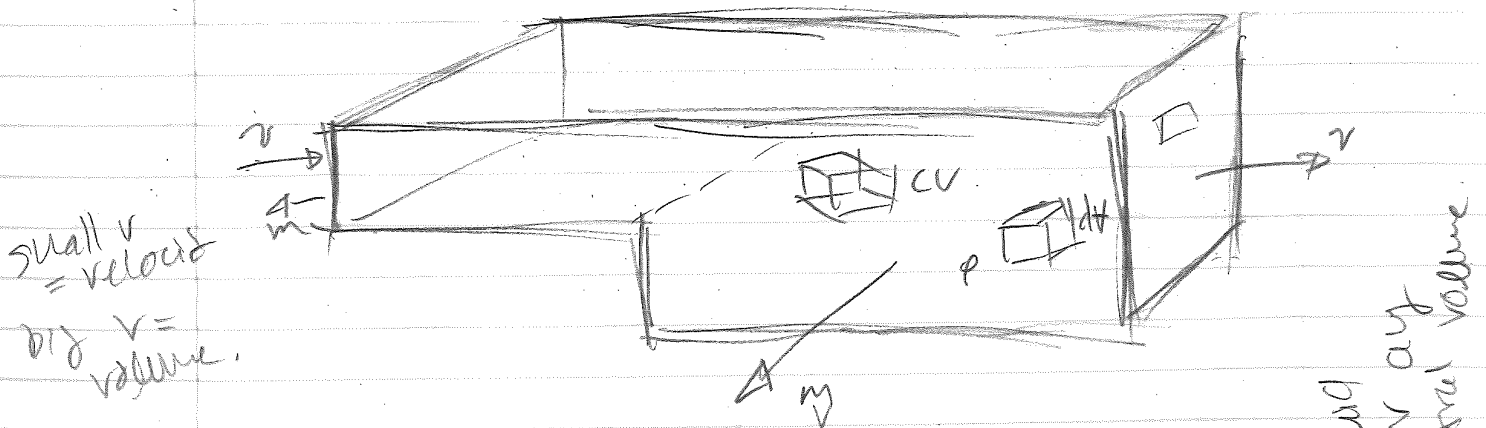
I/E Equation of motion

solve

- 1) Conservation of mass
- 2) momentum balance
- 3) Conservation of energy
- 4) fluid gas laws.

## 1) Conservation of mass

Consider a control volume fixed volume in space through which fluid flows.



$$\left[ \text{rate of increase of mass in CV} \right] = \left[ \text{net rate at which mass enters/leaves} \right]$$

$$\Rightarrow \frac{d}{dt} \int_V \rho dV = - \int_S \rho v_n dA$$

value negative to indicate outward movement

$dV$ : differential volume  
 $dx dy dz$  or  
 $dV d\theta dz$

$$dA = dx dy$$

$$\rho = \rho(\underline{x}, y, z, t) = \rho(\underline{x}, t) \quad \text{fluid density}$$

$\underline{x}$   
can write with position vector

$$\text{fluid velocity } \underline{v} = [u(\underline{x}, t), v(\underline{x}, t), w(\underline{x}, t)]$$

$\underline{n}$ : unit normal vector.

$\Rightarrow$  Consider small differential volume element:

$\Omega$  fixed in space  $\Rightarrow$  does not depend on time.

$$\hookrightarrow \frac{d}{dt} \iiint_{\Omega} \rho(\underline{x}, t) dV = \iiint_{\Omega} \frac{\partial \rho}{\partial t}(\underline{x}, t) dV$$

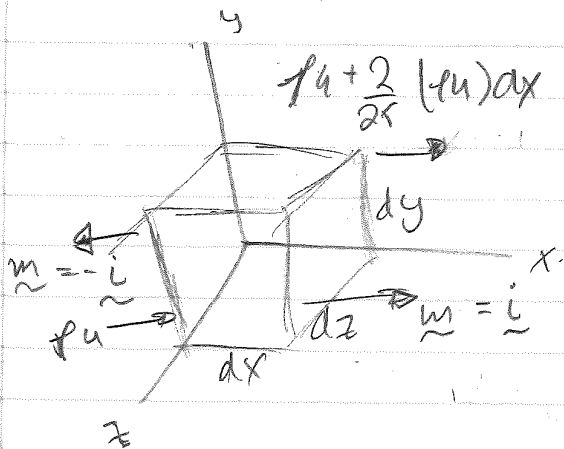
(Leibniz theorem)

$$\frac{\partial \rho}{\partial t} = \frac{\partial \rho}{\partial t} \Big|_{\underline{x} \text{ fix}} \quad \text{time dependent}$$

$t$  point  $\underline{x}$  fixed.

$\Omega$  is very small  $\Rightarrow \frac{\partial \rho}{\partial t}$  is constant.

$$\iiint \frac{\partial \rho}{\partial t} dV = \frac{\partial \rho}{\partial t} \Big|_{xyz} \quad dx dy dz$$



$\rho \left( u + \frac{2}{2x} (u) dx \right)$  the net mass flux has contribution from all 6 faces.

Surfaces moved along the x-direction  
left control surface.

$$-\iint_{CS} (\underline{v} \cdot \underline{m}) dA = + \rho u \Big|_x dz dy$$

CS  
control surface

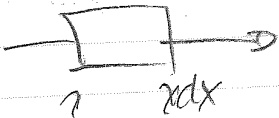
$$-\iint_{CS} (\underline{v} \cdot \underline{m}) dA = - \rho u \Big|_{x+dx} dz dy$$

↳ complete Taylor series expansion

$$\rho u \Big|_{x+dx} = \rho u \Big|_x + \frac{2}{2x} (\rho u)_x dx + \text{HOT}$$

The net contribution from L/R surfaces:

$$-\frac{2}{2x} (\rho u)_x dx dy dz + \text{HOT}$$

eg.   $\frac{\partial}{\partial x} (\rho u) > 0$

$$(\rho u)_{x+dx} > (\rho u)_x$$

then

$$\frac{d}{dt} (\dots) = -2$$

same argument for front + back faces.

$$-2 \int (\rho u) dz dy dz$$

top + bottom

$$-2 \int (\rho v) dx dy dz$$

$$\| \frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x} (\rho u) + \frac{\partial}{\partial y} (\rho v) + \frac{\partial}{\partial z} (\rho w) = 0$$

Differential form of the conservation of mass

$\equiv$  continuity equation

to find polar continuity see noodle pdf

introduce del operator  $\nabla \equiv$

$$\hat{i} \frac{\partial}{\partial x} + \hat{j} \frac{\partial}{\partial y} + \hat{k} \frac{\partial}{\partial z}$$

$$\frac{\partial p}{\partial t} + \nabla \cdot (p \vec{v}) = 0$$

Makes  
eq more  
compact.

new  
concept

Material derivative:  $\frac{D}{Dt} = \frac{\partial}{\partial t} + \vec{v} \cdot \nabla$

$$\nabla \cdot (p \vec{v}) = (\vec{v} \cdot \nabla) p + p (\nabla \cdot \vec{v})$$

show that  $\left| \frac{Dp}{Dt} + p (\nabla \cdot \vec{v}) = 0 \right|$

consider incompressible flow  $\rightarrow$  flow for which  
independent of  $x$  and time  $\rho = \text{constant}$ .

$$\Rightarrow \nabla \cdot \vec{v} = 0 \quad \text{or} \quad \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0$$

Mach number  $M = \frac{v}{a} < 0.2$

$v$  = characteristic velocity  
 $a$  = velocity of sound

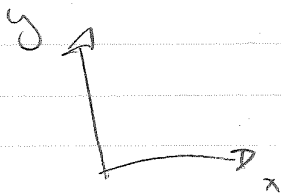
→ e.g. air  $a = 356 \text{ m/s} = 760 \text{ mph}$

$$M = 0.13$$

hurricane 150 mph  $M = 0.2$  marginal

→ e.g. 2D incompressible flow

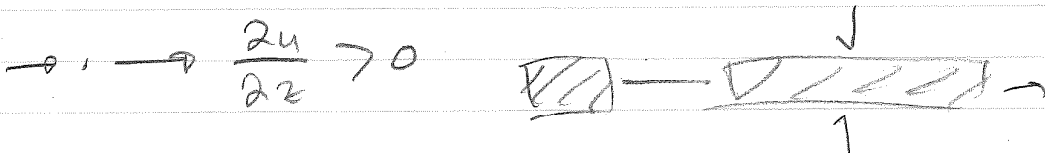
neglect  $z$  variation  $\left(\frac{\partial}{\partial z} = 0\right)$



$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \Rightarrow -\frac{\partial u}{\partial x} = \frac{\partial v}{\partial y} \text{ negative}$$

if  $\frac{\partial u}{\partial x} > 0$  locally then  $\frac{\partial v}{\partial y} < 0$

to conserve mass



→ e.g.  $u = \alpha x^2$  ( $\alpha > 0$ , constant)

$$\frac{\partial u}{\partial x} = 2\alpha x = -\frac{\partial v}{\partial y} \Rightarrow v = -2\alpha xy + f(x)$$

$$\frac{\partial \varphi}{\partial t} + \rho \nabla \cdot \vec{v} = 0$$

$$\varphi(x, y, z, t)$$

$$\nabla \cdot \vec{v} = 0$$

For 2D flow

$$(x, y) \\ (r, \theta)$$

Smits 6.3

also useful for stream lines

Incompressible.

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

introduce stream function  $\psi(x, y)$

definition

$$u = \frac{\partial \psi}{\partial y}(x, y)$$

$$v = -\frac{\partial \psi}{\partial x} \leftarrow \text{definition}$$

take continuity equation

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = \frac{\partial}{\partial x} \left( \frac{\partial \psi}{\partial y} \right) + \frac{\partial}{\partial y} \left( -\frac{\partial \psi}{\partial x} \right)$$

$$\frac{\partial^2 \psi}{\partial x \partial y} - \frac{\partial^2 \psi}{\partial y \partial x} = 0$$

as long as you  
are doing this  
don't have to check  
that continuity  
holds.

if  $\psi$  exists:

1) Solve for unknown instead of  $(u, v)$

2) Guarantee the conservation of mass.

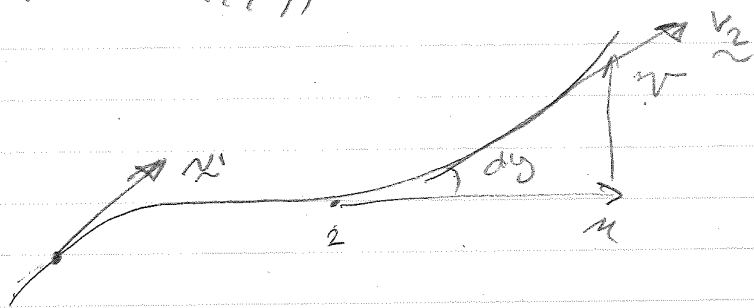
Streamline: line which is tangent to the velocity vector.

Streamline is at a fixed time  $(t)$ .

line // to the local velocity

$$\frac{dy}{dx} = \frac{v(x, y)}{u(x, y)}$$

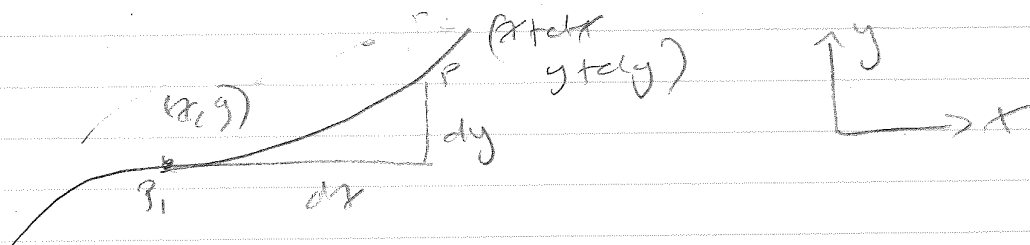
Smits 4.1.1



(chapter 4.1)

Consider 2D, incompressible such as  $\psi$  exist.

• 2 points close together along a streamline.





Taylor expansion

The value @ Pt  $\Psi(x+dx, y+dy) =$

$$\Psi(x, y) + \frac{\partial \Psi}{\partial x} dx + \frac{\partial \Psi}{\partial y} dy$$

$$= \Psi(x, y) - v dx + y dy$$

the streamline is such that

$$\frac{dy}{dx} = \frac{v}{u} \Rightarrow u dy = v dx$$

$$\Psi(x+dx, y+dy) = \Psi(x, y) \quad \text{then}$$

$$\Psi = \text{const}$$

if we can find two other  
we can just plot  
constant values of the  
streamline.

if  $\Psi(x, y)$  const,  $\Psi = \text{const} \equiv$  streamline

↳ in Matlab could do contour plot

The velocity can change right the stream  
function doesn't change.

↳ this is only about the direction of  
the field NOT the magnitude.

$$\text{e.g. } u = -\alpha x = \frac{\partial \psi}{\partial y}$$

$$v = +\alpha x = -\frac{\partial \psi}{\partial x}$$

→ integrate u over y

$$\psi(x, y) = \int u dy = \int -\alpha x dy$$

$$= -\alpha xy + f(x)$$

$$\frac{\partial (-\alpha xy + f(x))}{\partial y} = -\alpha x + 0$$

$$\psi(x, y) = -\int v dx$$

$$= -\int \alpha y dx$$

$$= -\alpha xy + g(y)$$

↓ put them together.

$$f(x) = g(y) = C_1$$

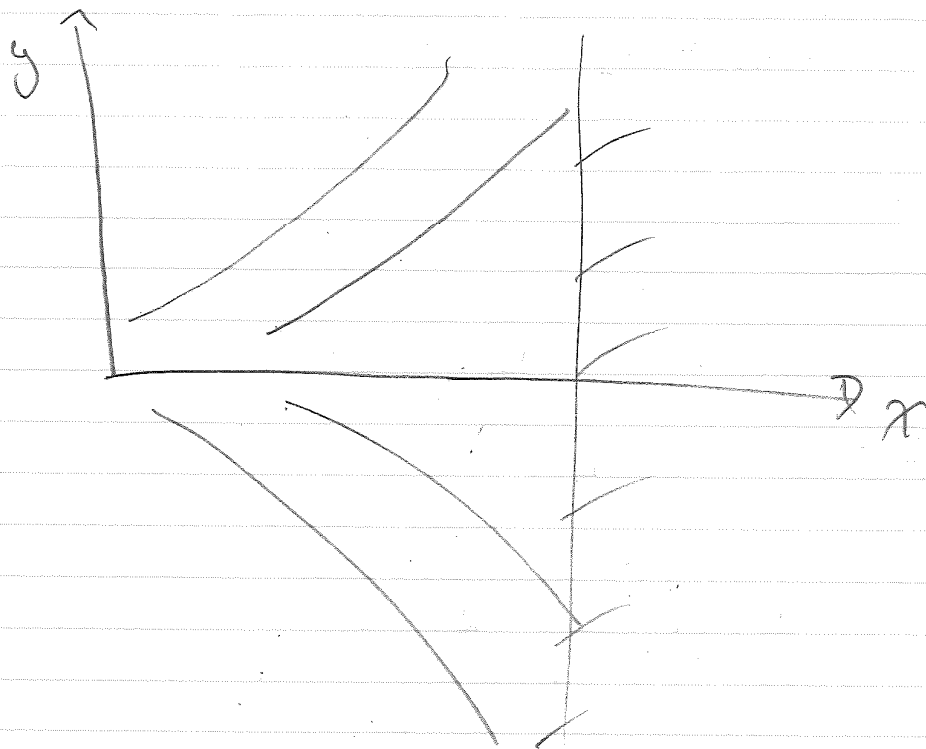
Streamline  $\psi = \text{const} = C_2$

$$-\alpha xy + C_1 = C_2$$

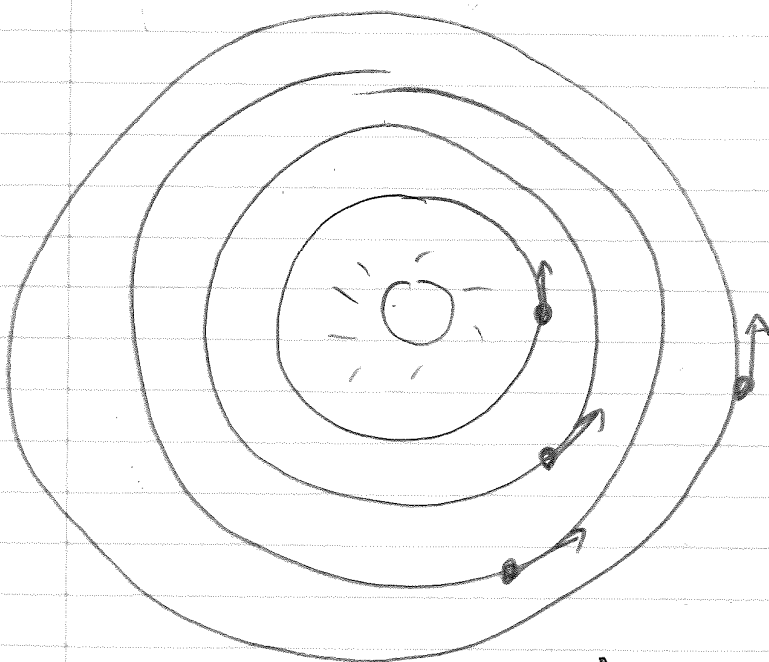
$$\alpha xy = C_1 - C_2 = C$$

$$y(x)$$

$$\boxed{y = \frac{C}{\alpha} \frac{1}{x}}$$



Recitation



$$\psi = 4$$

$$\psi = 3$$

$$\psi = 2$$

$$\psi = 1$$

[Streamlines don't cross]