

HW Assignment # 3

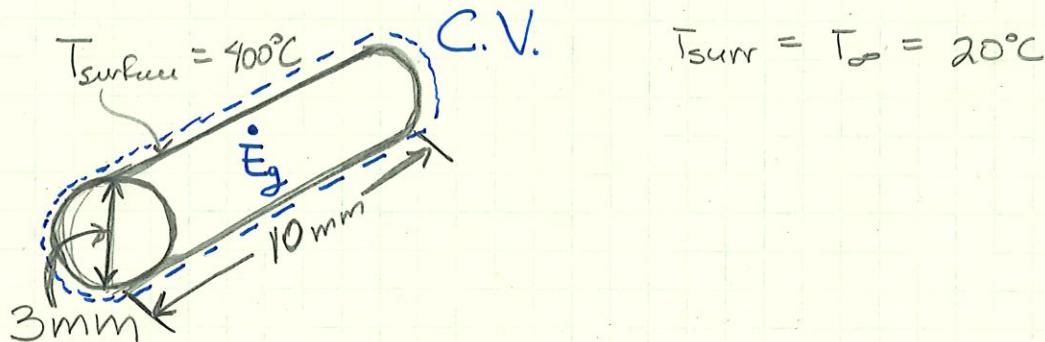
## Problem # 1

Statement: Steady-State soldering iron tip. Cylindrical with 3 mm diameter and 10 mm long.  $T_{\infty} = T_{\text{sur}} = 20^{\circ}\text{C}$ ,  $h = 20 \text{ W/m}^2\text{-K}$   
 $\epsilon_{\text{new}} \ll 1$ ,  $\epsilon_{\text{aged}} = 0.8$ ,  $T_{\text{surface}} = 400^{\circ}\text{C}$

Find: Required power for two cases:

- a) New tip,  $\epsilon \approx 0$
- b) Aged tip,  $\epsilon = 0.8$

## Schematic:



Assumptions:

- Constant properties, steady state.
- Heat loss from one end and peripheral surfaces.
- Uniform surface temperature and heat transfer coefficient.
- Aged tip is treated as a small gray surface in a large enclosure.
- For new tip, radiation is negligible.

Analysis

- Starting with Energy Balance

$$\cancel{E_{in}} - \cancel{E_{out}} + E_g = \cancel{\Delta E}_{ST} \quad \text{steady state}$$

$$E_g = E_{out}$$

$$E_g = \text{Power Supplied} = \dot{q}_c$$

$$E_{out} = q_{conv} + q_{radiation}$$

- First calculate  $E_{out}$ :

$$q_{conv} = h A_s (T_s - T_\infty)$$

$$A_s = 2\pi r L + \pi r^2$$

$$= 2\pi (1.5 \cdot 10^{-3} \text{ m})(0.01 \text{ m}) + \pi (1.5 \cdot 10^{-3} \text{ m})^2$$

$$= 1.013 \cdot 10^{-4} \text{ m}^2$$

$$q_{conv} = 20 \frac{\text{W}}{\text{m}^2 \text{K}} (1.013 \cdot 10^{-4} \text{ m}^2) (673 - 293) \text{ K}$$

$$= 0.769 \text{ W}$$

- Second calculate radiation for  $\epsilon = 0.8$

$$q_{radiation} = \epsilon \sigma A_s (T_s^4 - T_{surr}^4)$$

$$= 0.8 (5.67 \cdot 10^{-8} \frac{\text{W}}{\text{m}^2 \text{K}^4}) (1.013 \cdot 10^{-4} \text{ m}^2) (673^4 - 293^4) \text{ K}$$

$$= 0.9088 \text{ W}$$

## Problem # 1 continued:

- For new tip:

$$\dot{q}_{\text{new tip}} = \dot{q}_{\text{conv}}$$

$$\boxed{\dot{q}_{\text{new tip}} = 0.77 \text{ W}}$$

$$\dot{q}_{\text{new tip}} = \frac{\dot{q}_{\text{new tip}}}{\text{Volume}} = \frac{0.77 \text{ W}}{\pi (1.5 \cdot 10^{-3} \text{ m})^2 (0.01 \text{ m})}$$

$$\boxed{\dot{q}_{\text{new tip}} = 1.09 \cdot 10^7 \frac{\text{W}}{\text{m}^3}}$$

- For aged tip:

$$\dot{q}_{\text{aged tip}} = \dot{q}_{\text{conv}} + \dot{q}_{\text{radiation}} = 0.769 \text{ W} + 0.9088 \text{ W}$$

$$\boxed{\dot{q}_{\text{aged tip}} = 1.68 \text{ W}}$$

$$\boxed{\dot{q}_{\text{aged tip}} = 2.37 \cdot 10^7 \frac{\text{W}}{\text{m}^3}}$$

### Comments:

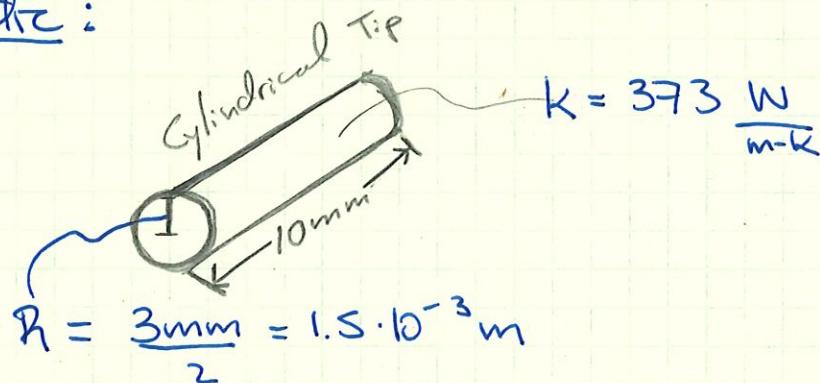
For the new polished tip radiation was neglected. The old tip has more than twice the power requirement for the new tip.

## Problem #2

Statement: Same steady-state tips (bath new and aged) as in problem #1.

Find: Steady-state temperature distributions for new and aged tips.

Schematic:



- Assumptions:
- Constant properties
  - 1-D in radial direction.
  - Steady-state
  - Soldering tip constructed from copper with  $k = 373 \frac{\text{W}}{\text{m}\cdot\text{K}}$  at an average temp of 700 K.

Analysis:

The Heat Diffusion Equation in cylindrical coordinates is:

$$\frac{1}{r} \frac{\partial}{\partial r} \left( kr \frac{\partial T}{\partial r} \right) + \frac{1}{r^2} \frac{\partial}{\partial \phi} \left( k \frac{\partial T}{\partial \phi} \right) + \frac{\partial}{\partial z} \left( k \frac{\partial T}{\partial z} \right) + q = \rho c \frac{\partial T}{\partial t}$$

Since we have assumed 1-D and steady state,

Problem #2 continued...

So we have:

$$\frac{1}{r} \frac{d}{dr} \left( kr \frac{dT}{dr} \right) + \dot{q} = 0$$

- Boundary conditions:

$$T|_{r=R} = 673 \text{ K} \quad (\text{outer surface})$$

$$\frac{dT}{dr} \Big|_{r=0} = 0 \quad (\text{center line adiabatic due to symmetry})$$

- First solve for new tip condition,  $\dot{q}_{\text{real}} = 0$   
Separate variables and integrate

Since  $k = \text{constant}$

$$\frac{k}{r} \frac{d}{dr} \left( r \frac{dT}{dr} \right) + \dot{q} = 0$$

$$k \frac{d}{dr} \left( r \frac{dT}{dr} \right) = -\dot{q} r$$

$$kr \frac{dT}{dr} = -\dot{q} \left( \frac{r^2}{2} \right) + C_1$$

$$\Rightarrow \frac{dT}{dr} = -\frac{\dot{q} r}{2k} + \frac{C_1}{kr}$$

Integrating again:

$$T(r) = -\frac{\dot{q} r^2}{4k} + C_2 + \frac{C_1}{k} \ln r$$

- Now apply B.C. at  $r=0$ :  $\frac{dT}{dr} \Big|_{r=0} = 0$

$$\frac{dT}{dr} \Big|_{r=0} = \left( -\frac{\dot{q} r}{2k} + \frac{C_1}{kr} \right) \Big|_{r=0} = \frac{C_1}{kr} \Big|_{r=0} = 0 \Rightarrow C_1 = 0$$

Problem #2 continued

- Now apply B.C. at  $r=R$

$$T \Big|_{r=R} = 673 \text{ K}$$

$$673 \text{ K} = -\frac{\dot{q} R^2}{4 \text{ K}} + C_2$$

$$\Rightarrow C_2 = 673 \text{ K} + \frac{\dot{q} R^2}{4 \text{ K}}$$

$$\Rightarrow T(r) = -\frac{\dot{q} r^2}{4 \text{ K}} + \frac{\dot{q} R^2}{4 \text{ K}} + 673 \text{ K}$$

$$T(r) = \frac{\dot{q}}{4 \text{ K}} (R^2 - r^2) + 673 \text{ K}$$

- Substituting  $\dot{q}$  for the new tip (convection only)

$$T(r) = 1.09 \cdot 10^7 \frac{\text{W}}{\text{m}^3} (R^2 - r^2) + 673 \text{ K}$$

Where we used  $\dot{q}_{\text{new tip}}$  from problem #1

- Substituting  $\dot{q}$  for the old tip ( $\dot{q}_{\text{old tip}}$  from problem #1)

$$T(r) = \frac{2.37 \cdot 10^7 \frac{\text{W}}{\text{m}^3}}{4 \text{ K}} (R^2 - r^2) + 673 \text{ K}$$

Problem # 2 continued

- The maximum temperature occurs at  $r=0$ .  
Thus, for the new tip:

$$\begin{aligned} T_{\max \text{ new}} &= \frac{\dot{q}}{4k} (R^2 - r^2)^0 + 673 \text{ K} = \frac{\dot{q} R^2}{4k} + 673 \text{ K} \\ &= \frac{1.09 \cdot 10^7 \frac{\text{W}}{\text{m}^3}}{4.373 \frac{\text{W}}{\text{mK}}} (1.5 \cdot 10^{-3} \text{ m})^2 + 673 \text{ K} \\ &= 0.01621 \text{ K} + 673 \text{ K} \approx 673 \text{ K} = 400^\circ \text{C} \end{aligned}$$

$T_{\max \text{ new}} = 400^\circ \text{C}$

Answer

For the old tip:

$$\begin{aligned} T_{\max \text{ old}} &= \frac{2.37 \cdot 10^7 \frac{\text{W}}{\text{m}^3}}{4.373 \frac{\text{W}}{\text{mK}}} (1.5 \cdot 10^{-3} \text{ m})^2 + 673 \text{ K} \\ &= 0.0357 \text{ K} + 673 \text{ K} \approx 673 \text{ K} = 400^\circ \text{C} \end{aligned}$$

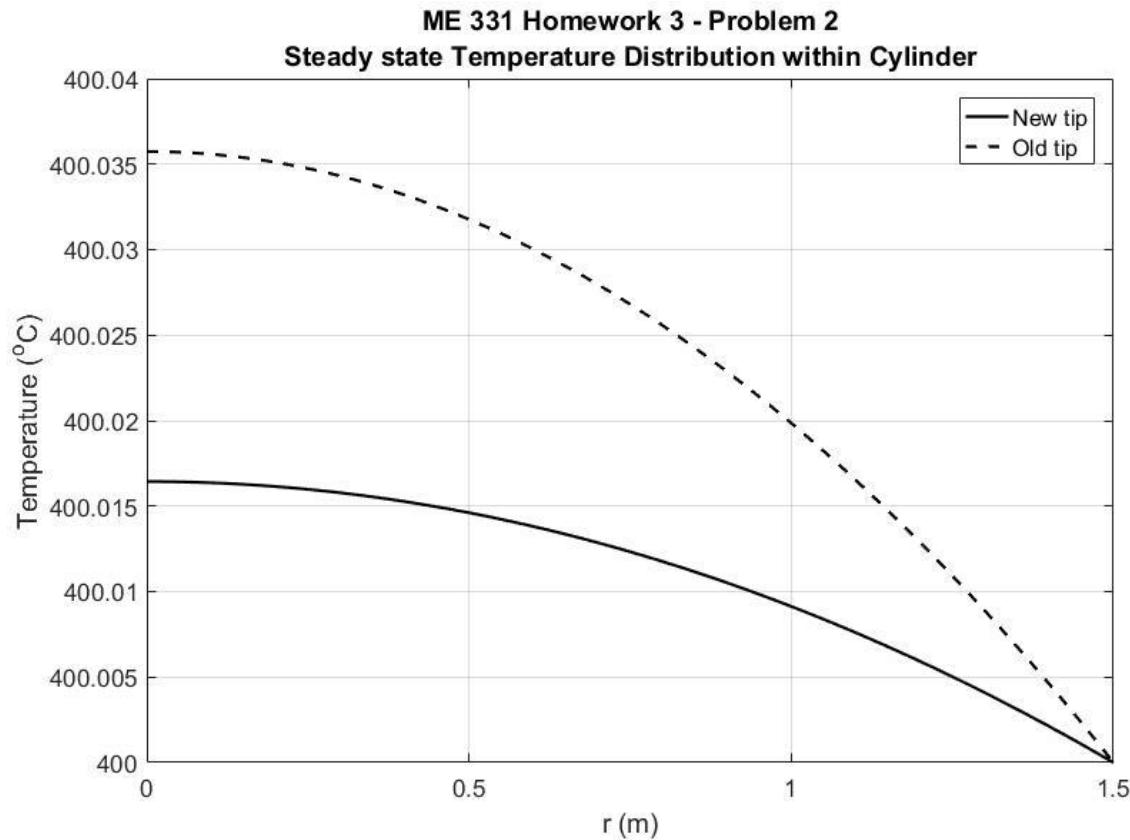
$T_{\max \text{ old}} = 400^\circ \text{C}$

Answer

See attached plots and MATLAB code.

Comments: Due to negligible conduction resistance, there are extremely small thermal gradients within the soldering iron tips, whether for the new or aged cases.

Problem #2 continued



```

%% Plot temperature profiles for new and old tips

r = 0:1e-5:1.5e-3;
R = 1.5e-3; % m
k = 373; %W/m-K

% New tip - convection only
qnew = 1.09e7; %W/m^3
Tnew = (qnew/(4*k)) * (R^2 - r.^2) + 673;

% Old tip - convection plus radiation
qold = 2.37e7; %W/m^3
Told = (qold/(4*k)) * (R^2 - r.^2) + 673;

figure(1); clf
plot(1000*r,Tnew-273,'k','Linewidth',1.6); grid on; hold on
plot(1000*r,Told-273,'k--','Linewidth',1.6);
xlabel('r (m)');
ylabel('Temperature (^oC)')
legend('New tip','Old tip')

title({'ME 331 Homework 3 - Problem 2'; 'Steady state Temperature Distribution within Cylinder'})
set(gca,'Fontsize',13)

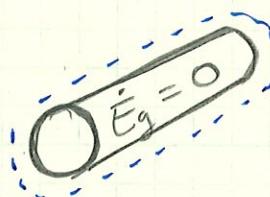
```

### Problem 3

Statement: Soldering iron tips (both new and aged) operating at steady state, then turned off.

Find: Temperature within tip as a function of time for both cases.

Schematic:



C.V. Lumped!

Assumptions:  $T_{\text{initial}} = 673 \text{ K}$

- Negligible temperature changes within tips, Apply lumped capacitance model.
- Constant properties at  $T = 400 \text{ K}$   
 $\rho = 8933 \text{ kg/m}^3$ ;  $k = 393 \frac{\text{W}}{\text{mK}}$ ;  $c = 397 \frac{\text{J}}{\text{kgK}}$   
 $h = 20 \text{ W/(m}^2\text{K)}$

Analysis

- Check  $Biot \# = Bi = \frac{hL_c}{k}$

Where ~~L~~  $L_c = \frac{\text{Volume}}{\text{Surface Area}} = \frac{\pi r^2 L}{\pi r^2 + 2\pi r L} \approx \cancel{2\pi r L} = 6.9767 \cdot 10^{-4} \text{ m}$

$$Bi = \frac{20 \frac{\text{W}}{\text{m}^2\text{K}} \cdot 6.9767 \cdot 10^{-4} \text{ m}}{393 \frac{\text{W}}{\text{mK}}} = 3.5 \cdot 10^{-5} \ll 0.1$$

So lumped works!

Problem 3 continued:

- Now apply conservation of Energy to lumped C.V.

$$\dot{E}_{\text{stored}} = -\dot{E}_{\text{out}}$$

$$\rho V_c \frac{dT}{dt} = -h A_s (T - T_{\infty}) \quad \text{for new } t_{\text{tip}}$$

$$\Rightarrow \int_0^t \frac{-h A_s}{\rho V_c} dt = \int_{T_i}^T \frac{1}{T - T_{\infty}} dT$$

$$-\frac{h A_s}{\rho V_c} t = \ln \left( \frac{T - T_{\infty}}{T_i - T_{\infty}} \right)$$

$$\Rightarrow \boxed{\frac{T - T_{\infty}}{T_i - T_{\infty}} = \exp \left( -\frac{h A_s}{\rho V_c} t \right)} \quad \text{For new } t_{\text{tip}}$$

- For the aged tip we have to account for radiation as well.

We get

$$\frac{dT}{dt} = -\frac{A_s}{\rho V_c} [h(T - T_{\infty}) + \epsilon \sigma (T^4 - T_{\text{sur}}^4)]$$

Where  $T_{\text{sur}} = T_{\infty} = 20^{\circ}\text{C} = 293\text{ K}$

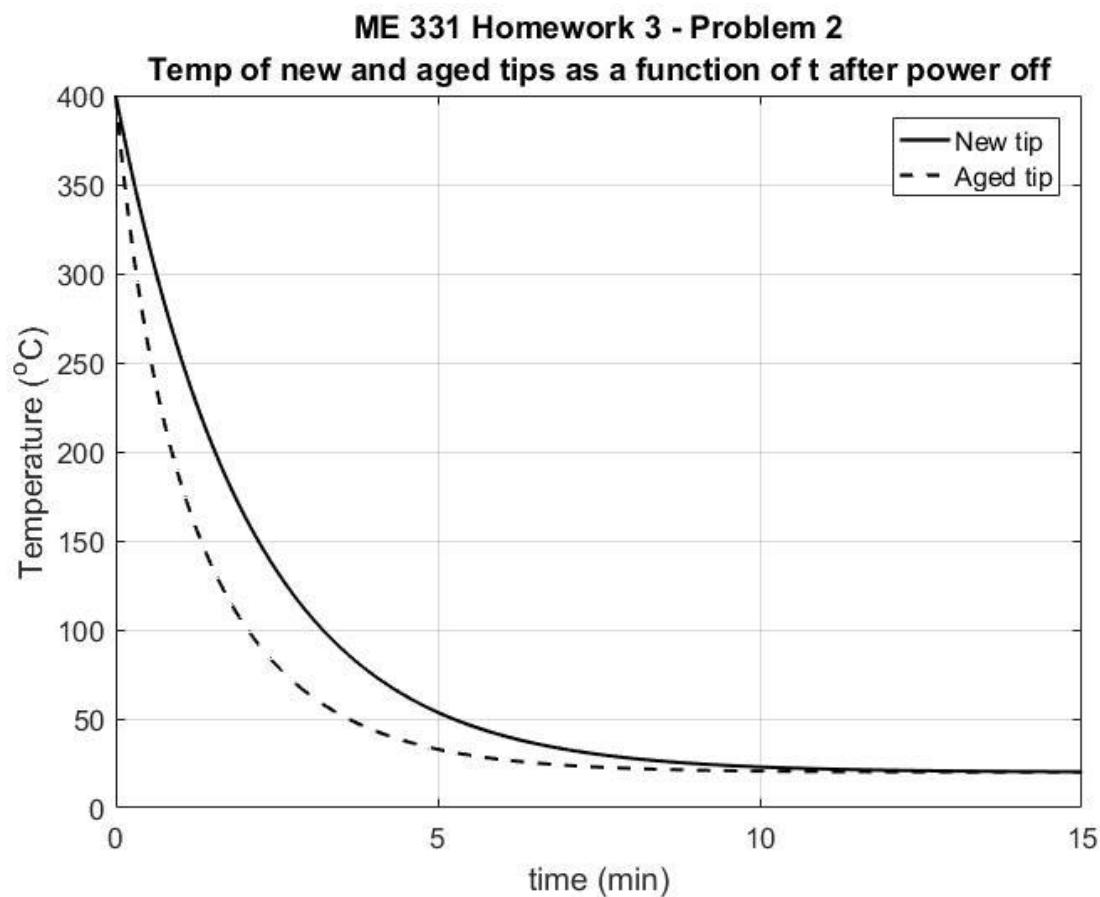
$\epsilon = 0.8$  and we can solve numerically  
For a sufficiently small time step we can write

$$\frac{dT}{dt} \approx \frac{\Delta T}{\Delta t}$$

For temperatures at times  $t_1, t_2, t_3, \dots, t_n, t_n, t_{n+1}, \dots$

$$T_{n+1} = T_n + \Delta t \left\{ -\frac{A_s}{\rho V_c} [h(T - T_{\infty}) + \epsilon \sigma (T^4 - T_{\text{sur}}^4)] \right\}$$

Problem #3 continued



Comments:

The aged tip cools more rapidly due to radiation.

(see MATLAB code below)

### Problem #3 continued

```

%% ME331 : Homework 3
%% P3

r = 1.5e-3; L = 0.01; % m
k = 393; c = 397;
V = pi*(r^2)*L; A = pi*r^2 + 2*pi*r*L;
Lc = V/A

h = 20;

Bi = h*Lc/k

rho = 8933; sigma = 5.67e-8; epsilon = 0.8;
Tinf = 293; To = 673;
%% Plot new tip Temps
t = 0:2:1000;
T = Tinf+(To-Tinf)*exp(-(h*A/(rho*V*c))*t);

figure(1); clf
plot(t/60,T-273,'k','Linewidth',1.6); grid on; hold on

%% Plot old tip temps
const = -A/(rho*V*c);
fun = @(T)(const*(h*(T-Tinf) + epsilon*sigma*(T^4-Tinf^4)) );

deltat = 1e-2; %small time step 1/100 s
t = 0:deltat:1000;
T = zeros(size(t));
T(1) = 673; %initial condition
for k = 2:length(t)
    T(k) = T(k-1) + deltat*fun(T(k-1));
end

plot(t/60,T-273,'k--','Linewidth',1.6);
xlabel('time (min)'); ylabel('Temperature (^oC)')
legend('New tip','Aged tip')

title({'ME 331 Homework 3 - Problem 2'; ...
    'Temp of new and aged tips as a function of t after power off'})
set(gca,'Fontsize',13)

xlim([0 15])

```

## Problem # 4

Statement : Water main is positioned below ground to avoid freezing. Initial soil temperature is  $20^{\circ}\text{C}$ , and ground surface temperature is  $-25^{\circ}\text{C}$  for 100 days.

- Ground - soil  $T_i = 20^{\circ}\text{C}$
- Avoid freezing  $\text{H}_2\text{O}$  main,  $T > 0^{\circ}\text{C}$
- $T_s$  of soil =  $-25^{\circ}\text{C}$  for 100 days.

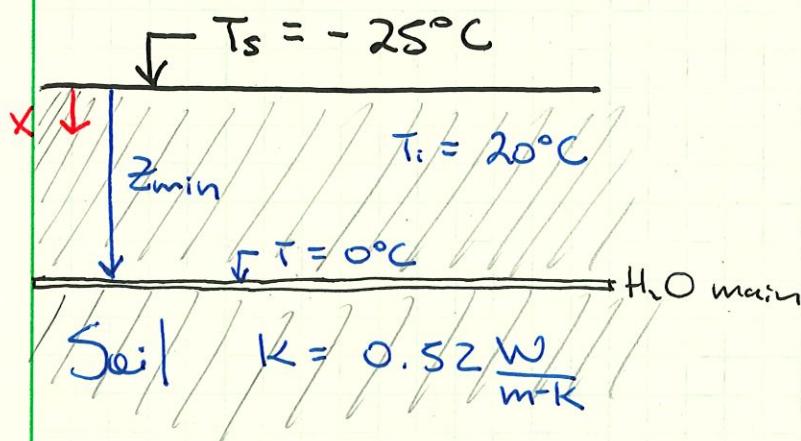
Find : - Minimum depth ( $z_{\min}$ ) for  $\text{H}_2\text{O}$  main to avoid freezing.

$$\Rightarrow T(z_{\min}, 100 \text{ days}) = 0^{\circ}\text{C}$$

Assumptions :

- Soil is a semi-infinite medium
- Constant surface temp.
- Constant properties

### Schematic :



$$k_{\text{soil}} = 0.52 \frac{\text{W}}{\text{mK}}$$

$$\rho_{\text{soil}} = 2050 \frac{\text{kg}}{\text{m}^3}$$

$$c_{\text{soil}} = 1840 \frac{\text{J}}{\text{kgK}}$$

$$\lambda = \frac{k}{\rho c} = \frac{0.52 \frac{\text{W}}{\text{mK}}}{2050 \frac{\text{kg}}{\text{m}^3} \cdot 1840 \frac{\text{J}}{\text{kgK}}} = 1.3786 \cdot 10^{-7} \frac{\text{m}^2}{\text{W}}$$

## Problem 4 continued

Analysis:

- Semi infinite medium with constant  $T_s$

Since we have semi infinite medium with constant  $T_s$ , equation (5.60) from (Incropera, 2011) applies:

$$\frac{T - T_s}{T_i - T_s} = \operatorname{erf} \gamma \quad \text{with} \quad \gamma = \frac{x}{(4\alpha t)^{1/2}}$$

$$\Rightarrow \frac{T(z_{\min}, 100 \text{ days}) - T_s}{T_i - T_s} = \operatorname{erf} \left( \frac{z_{\min}}{2\sqrt{\alpha t}} \right)$$

$$\operatorname{erf} \left( \frac{z_{\min}}{2\sqrt{\alpha t}} \right) = \frac{0 - (-25)}{20 - (-25)} K = \frac{25}{45} = \frac{5}{9} = 0.5556$$

$$\text{Let } \omega = \frac{z_{\min}}{2\sqrt{\alpha t}} \Rightarrow \operatorname{erf}(\omega) = \frac{5}{9}$$

Interpolating in table B. 2

$$\omega = 0.5407 = \frac{z_{\min}}{2\sqrt{\alpha t}} \Rightarrow z_{\min} = 2 \cdot 0.5407 \sqrt{\alpha t}$$

$$z_{\min} = 2 \cdot 0.5407 \cdot \sqrt{1.3786 \cdot 10^{-3} \frac{\text{m}^2}{\text{s}}} \cdot 100 \cdot 3600 \cdot 24 \text{ s}$$

$$= 1.1803 \text{ m}$$

Rounding up to ensure no freezing possibility with  $-25^\circ\text{C}$  surface temperature for as long as 100 days.

$$z_{\min} = 1.2 \text{ m}$$

## Problem # 5

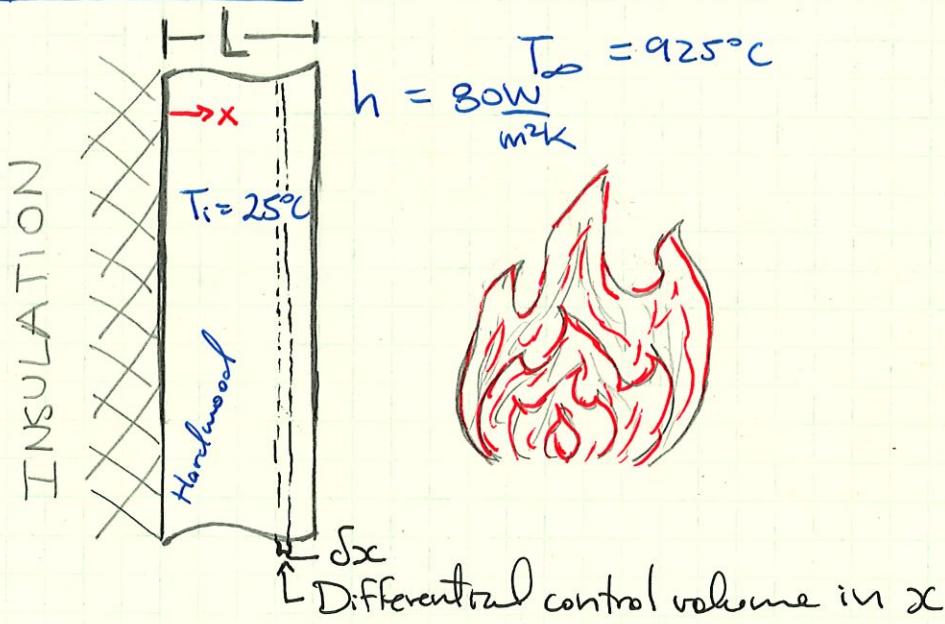
Statement: Fire door (planar solid hardwood) insulated on interior surface. 4.0 cm thick and with uniform initial temperature  $T_i = 25^\circ\text{C}$ . Test surface of door opposing the insulated surface by exposing to ambient temp of  $925^\circ\text{C}$  with  $h_{\text{eff}} = 80 \frac{\text{W}}{\text{m}^2\text{K}}$

Find: Temperature distribution as a function of time. When is single term approximation inaccurate? Show improvement by adding terms of the series.

- Plot temperature distribution as a function of time
- Range of times when single term approximation is inaccurate
- Using single-term approximation, Find  $T(x, t)$  and plot for different times  $t$ .
- Plot single term approximation for  $F_o < 0.2$ , compare to many term approx.

### Assumptions:

- Perfect insulation @  $x=0 \rightarrow$  Fire door is half of planar wall,  $L = 4.0 \text{ cm}$
- Constant properties
- 1-D conduction in  $x$ -direction

Schematic:

Hardwood at 300K - Table A.3

$$\rho = 720 \frac{\text{kg}}{\text{m}^3} \quad c = 1255 \frac{\text{J}}{\text{kg}\text{K}}$$

$$k = 0.16 \frac{\text{W}}{\text{m}\cdot\text{K}} \quad \alpha = \frac{k}{\rho c} = \frac{0.16 \frac{\text{W}}{\text{m}\cdot\text{K}}}{720 \frac{\text{kg}}{\text{m}^3} \cdot 1255 \frac{\text{J}}{\text{kg}\text{K}}} \\ \alpha = 1.77 \cdot 10^{-3} \frac{\text{m}^2}{\text{s}}$$

Analysis:

Since we assume perfect insulation on one side we can assume the door behaves as a half planar wall (see for instance Example 5.5 in textbook)

Hence we have:

$$\frac{T - T_o}{T_i - T_o} = \Theta^* = \sum_{n=1}^{\infty} C_n \exp(-\xi_n^2 F_o) \cos(\xi_n x^*) \quad (5.42a)$$

(5.34)

with  $x^* = \frac{x}{L}$ ,  $F_o = \frac{\alpha t}{L^2}$

$$C_n = \frac{4 \sin \zeta_n}{2\zeta_n + \sin(2\zeta_n)} \quad (5.42b)$$

~~$$\zeta_n \tan \zeta_n = Bi \quad (5.42c)$$~~

$$Bi = \frac{hL}{k}$$

We can check  $Bi$ :

$$Bi = \frac{hL}{k} = \frac{80 \frac{W}{m^2 K} \cdot 0.04m}{0.16 W/m \cdot K} = 20 > 0.1$$

Clearly we cannot use lumped capacitance.

- Single term approximation is valid when  $F_0 > 0.2$

$$F_0 > 0.2, F_0 = \frac{\alpha t}{L^2} \Rightarrow t > \frac{0.2 L^2}{\alpha}$$

$$t > \frac{0.2 \cdot (0.04m)^2}{1.77 \cdot 10^{-7} m^2/s} = 1807.2 s$$

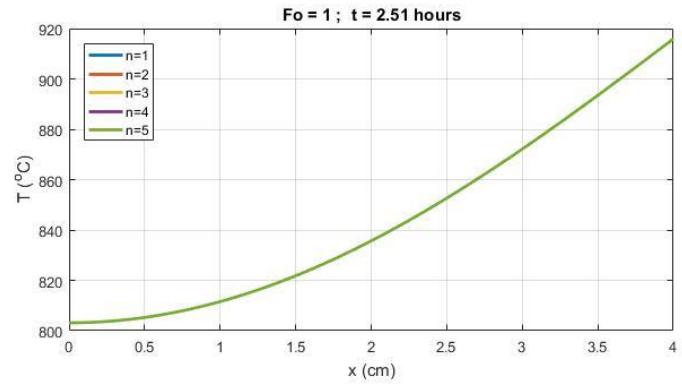
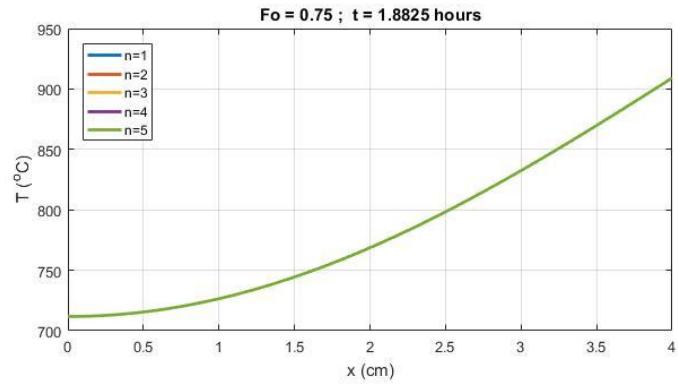
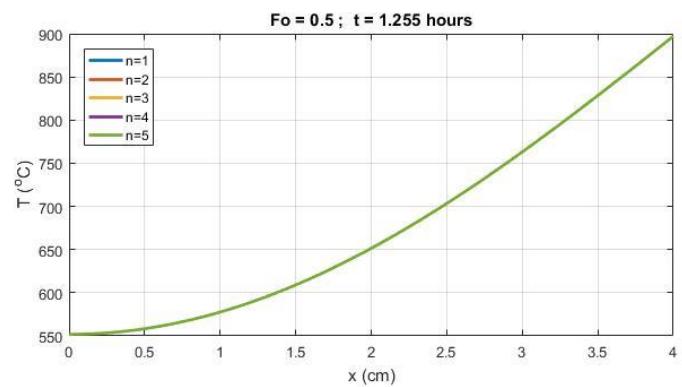
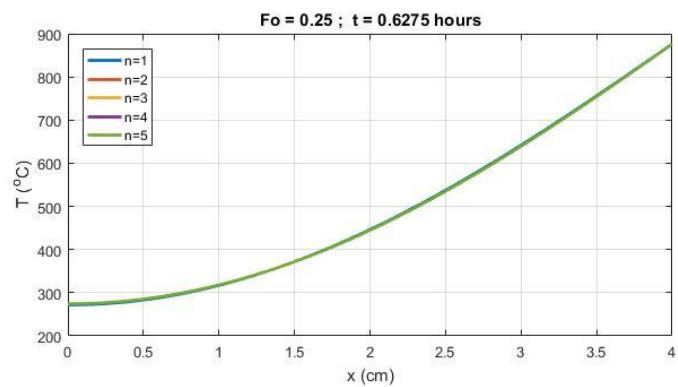
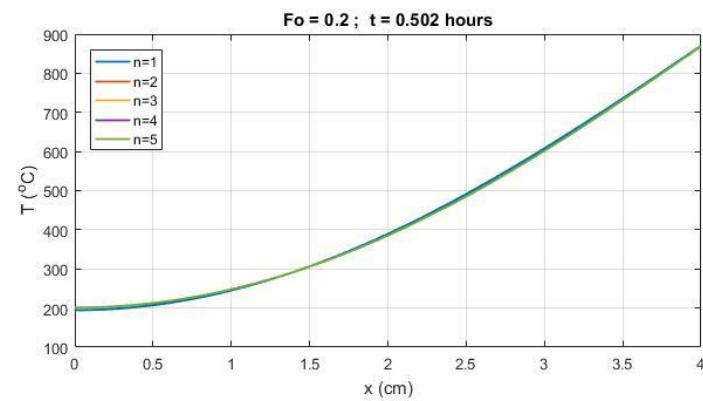
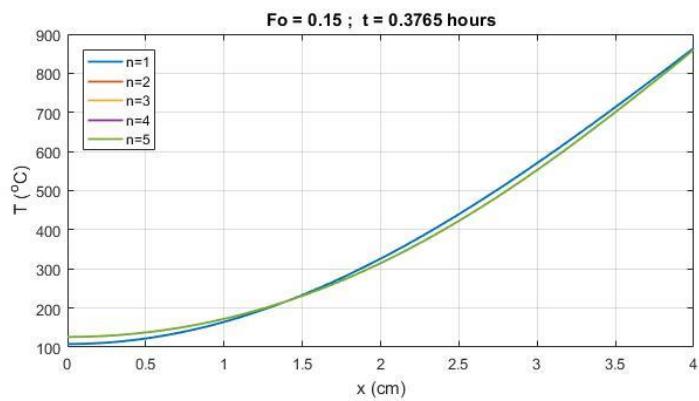
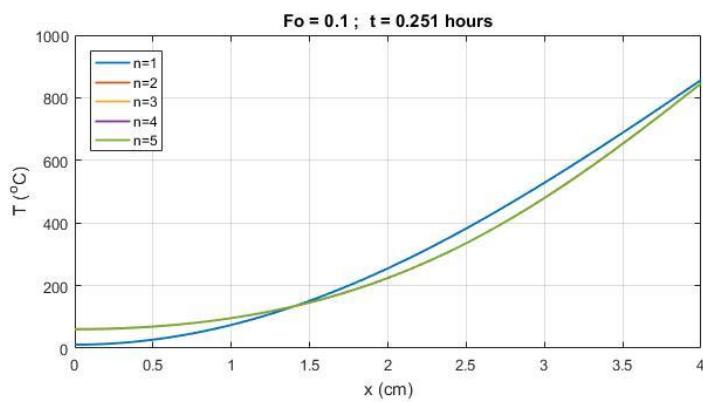
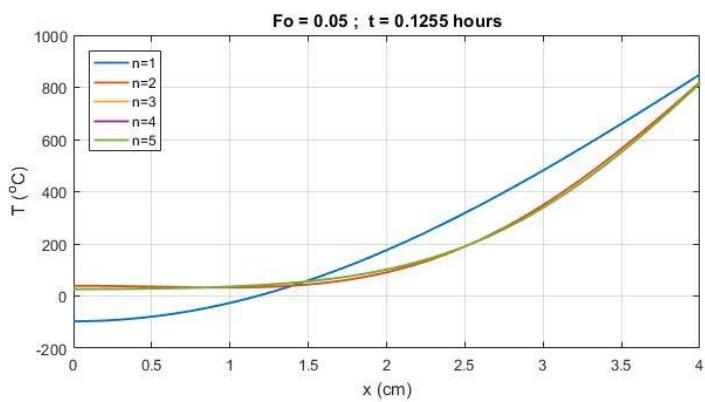
Thus

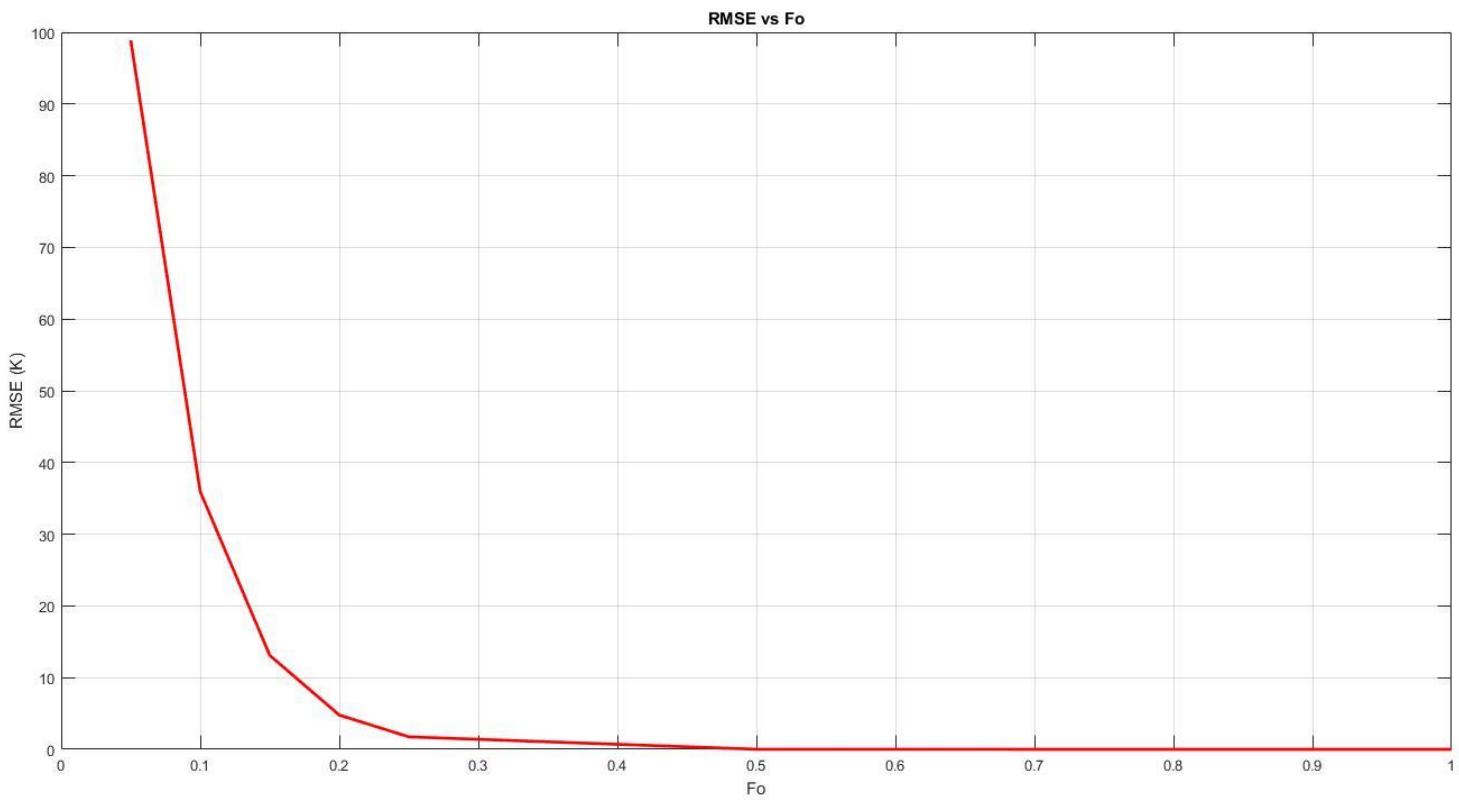
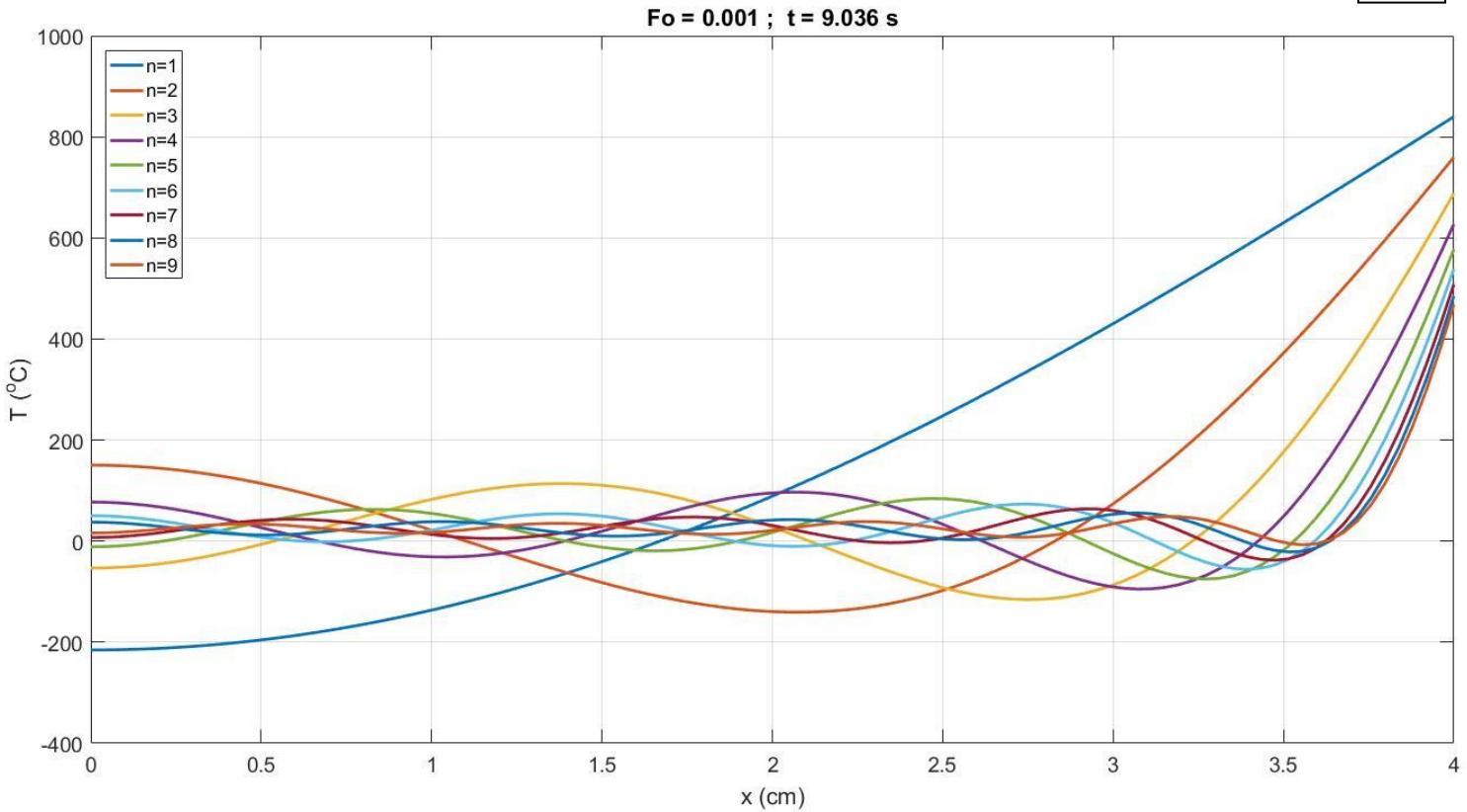
Single term approx is valid for  $t > 1800s$   
is inaccurate for  $0 < t < 1800s$

For temp. profile we have:

$$T = T_\infty + (T_i - T_\infty) \sum_{n=1}^{\infty} C_n e^{-(\zeta_n^2 F_0)} \cos(\zeta_n x^*)$$

See plots and code below.





RMSE computed from the error between the 1<sup>st</sup> term and five terms approximations.

**Comments:**

We see that for large enough values of the Fo number the series converges pretty quickly (as seen in the RMSE plot above). However the series does not converge quickly early in the process (at small t, small Fo). For instance, as appreciated in the plot for Fo = 0.001 it is evident that we need many more terms in the series to obtain an accurate solution.

**MATLAB code:**

```
clear; clc

%% ME331 : Homework 3

%% Problem 4
% properties
rho = 2050; k = 0.52; c = 1840; alpha = k/(rho*c);
L = 4e-2;
% solution of eqn 5.60
fun = @(w) (erf(w)-5/9);
w = fzero(fun,0.54);
Zmin = 2*w*sqrt(alpha*100*3600*24); % Zmin in m

%% Problem 5
% Properties
rho = 720; k = 0.16; c = 1255; alpha = k/(rho*c)
h = 80; % W/(m^2-K)

% for valid one term approx
tone = 0.2*(L^2)/alpha % t in s

%%%
% find the zeta_n and Cn
Bi = h*L/k

fun = @(x) (x.*tan(x)-Bi);
pnn = 1;
aold = -2;
zeta = [];
for knn = 0.3:0.5:30
    a = fzero(fun,knn);

    if abs(fun(a))<1e-9
        if abs(a-aold)>1e-6
            zeta(pnn) = a;
            pnn = pnn+1;
            aold = a;
        end
    end
end
end
```

```

zeta = unique(zeta);
zeta = zeta'; % i like column vectors
Cn = 4*sin(zeta)./(2*zeta+sin(2*zeta));

[zeta Cn]

Tinf = 925+273.15;
To = 25+273.15;
x = linspace(0,0.04);

%% PLOT first inaccurate
krm = 1;
figure(1); clf
for FoM = 1:4
    Fo = 0.05*FoM; t = (L^2)*Fo/alpha;
    for nterms = 1:5
        T = Temp_sol(x,t,zeta,Cn,To,Tinf,L,alpha,nterms);
        eval(['subplot 22' num2str(FoM)])
        plot(x*100,T-273.15,'Linewidth',1.4); hold on
    end
    xlabel('x (cm)'); ylabel('T (^oC)'); grid on
    title(['Fo = ' num2str(Fo) ' ; t = ' num2str(t/3600) ' hours'])
    legend('n=1','n=2','n=3','n=4','n=5','Location','Northwest')
    % get RMSE
    Fofo(krm) = Fo;
    RMSE(krm) = sqrt(mean((T -
Temp_sol(x,t,zeta,Cn,To,Tinf,L,alpha,1)).^2)); % Root Mean Squared Error
    krm = krm+1;
end

%% PLOT accurate
figure(2); clf
for FoM = 1:4
    Fo = 0.25*FoM; t = (L^2)*Fo/alpha;
    for nterms = 1:5
        T = Temp_sol(x,t,zeta,Cn,To,Tinf,L,alpha,nterms);
        eval(['subplot 22' num2str(FoM)])
        plot(x*100,T-273.15,'Linewidth',2); hold on
    end
    xlabel('x (cm)'); ylabel('T (^oC)'); grid on
    title(['Fo = ' num2str(Fo) ' ; t = ' num2str(t/3600) ' hours'])
    legend('n=1','n=2','n=3','n=4','n=5','Location','Northwest')
    Fofo(krm) = Fo;
    RMSE(krm) = sqrt(mean((T -
Temp_sol(x,t,zeta,Cn,To,Tinf,L,alpha,1)).^2)); % Root Mean Squared Error
    krm = krm+1;
end

[Fofo' RMSE']

%% for very small Fo

```

```

figure(3); clf
Fo = 0.001; t = (L^2)*Fo/alpha;
for nterms = 1:9
    T = Temp_sol(x,t,zeta,Cn,To,Tinf,L,alpha,nterms);
    plot(x*100,T-273.15,'Linewidth',2); hold on
end
xlabel('x (cm)'); ylabel('T (^oC)'); grid on
title(['Fo = ' num2str(Fo) ' ; t = ' num2str(t) ' s'])
legend('n=1','n=2','n=3','n=4','n=5','n=6','n=7','n=8','n=9',...
'Location','Northwest')
set(gca,'Fontsize',14)

figure(4); clf
plot(Fo, RMSE, 'r', 'Linewidth', 2); xlabel('Fo'); ylabel('RMSE (K)')
grid on
title('RMSE vs Fo')

function [ T ] = Temp_sol(x,t,zeta,Cn,To,Tinf,L,alpha,nterms)
%UNTITLED2 Summary of this function goes here
% Detailed explanation goes here
m = length(x) ;
theta = zeros(size(x));
zeta = zeta(1:nterms);
Cn = Cn(1:nterms);
Fo = alpha*t/(L^2);
for km = 1:m
    theta(km) = sum(Cn.*exp(-(zeta.^2)*Fo).*cos(zeta.*x(km)./L));
end

T = Tinf+(To-Tinf)*theta;

end

```