ME500/AA535 Advanced Composite Structural Analysis

A brief review of Chapters 1-7

(particularly Chapter 4-7)

Hooke's Law for an Anisotropic Material (neglecting thermal or moisture effects)

$$\varepsilon_{ij} = S_{ijkl}\sigma_{kl}, \quad i, j, k, l = x, y, z$$
(or)
$$\sigma_{ij} = C_{ijkl}\varepsilon_{kl}, \quad i, j, k, l = x, y, z$$

where:

 $\sigma_{ij}, \varepsilon_{ij} = 2$ nd order stress and strain tensors, respectively $S_{ijkl} = 4$ th order compliance matrix $C_{ijkl} = 4$ th order stiffness matrix

Would like to write Hooke's law using 2-D arrays, but cannot because of 4th order compliance and stiffness tensors
In the absence of body forces, stress and strain tensors are symmetric...allows use of "contracted notation"

$\varepsilon_{xx} \to \varepsilon_1$	$\sigma_{xx} \rightarrow \sigma_1$
$\varepsilon_{yy} \rightarrow \varepsilon_2$	$\sigma_{yy} \rightarrow \sigma_2$
$\mathcal{E}_{ZZ} \to \mathcal{E}_{3}$	$\sigma_{zz} ightarrow \sigma_3$
$\gamma_{yz} = \gamma_{zy} \to \varepsilon_4$	$\sigma_{yz} = \sigma_{zy} \rightarrow \sigma_4$
$\gamma_{xz} = \gamma_{zx} \to \varepsilon_5$	$\sigma_{xz} = \sigma_{zx} \rightarrow \sigma_5$
$\gamma_{xy} = \gamma_{yx} \to \varepsilon_6$	$\sigma_{xy} = \sigma_{yx} \rightarrow \sigma_6$

Hooke's Law for an Anisotropic Material (neglecting thermal or moisture effects)

$$\varepsilon_i = S_{ij}\sigma_j, \quad i, j = 1...6$$
(or)
$$\sigma_i = C_{ij}\varepsilon_j, \quad i, j = 1...6$$

where:

 σ_i , \mathcal{E}_i = (disguised) 2nd order stress and strain tensors, respectively S_{ij} = (disguised) 4th order compliance matrix C_{ij} = (disguised) 4th order stiffness matrix

$$\varepsilon_i = S_{ij}\sigma_j, \quad i, j = 1...6$$

$\left[\mathcal{E}_{1} \right]$	$\int S_{11}$	<i>S</i> ₁₂	<i>S</i> ₁₃	<i>S</i> ₁₄	<i>S</i> ₁₅	S_{16}	$[\sigma_1]$
$ \mathcal{E}_2 $	<i>S</i> ₂₁	<i>S</i> ₂₂	<i>S</i> ₂₃	<i>S</i> ₂₄	S ₂₅	S ₂₆	$ \sigma_2 $
$\left \varepsilon_{3} \right _{-}$	<i>S</i> ₃₁	<i>S</i> ₃₂	S ₃₃	S ₃₄	S ₃₅	S ₃₆	σ_3
\mathcal{E}_4	<i>S</i> ₄₁	S_{42}	<i>S</i> ₄₃	S_{44}	S_{45}	S ₄₆	σ_4
\mathcal{E}_5	<i>S</i> ₅₁	<i>S</i> ₅₂	<i>S</i> ₅₃	S_{54}	S ₅₅	S ₅₆	σ_5
$\left[\mathcal{E}_{6} \right]$	$\lfloor S_{61}$	S_{62}	S ₆₃	S_{64}	S_{65}	S_{66}	$\left[\sigma_{6}\right]$

$$\begin{cases} \boldsymbol{\varepsilon}_{xx} \\ \boldsymbol{\varepsilon}_{yy} \\ \boldsymbol{\varepsilon}_{zz} \\ \boldsymbol{\gamma}_{yz} \\ \boldsymbol{\gamma}_{xz} \\ \boldsymbol{\gamma}_{xy} \end{cases} = \begin{bmatrix} S_{11} & S_{12} & S_{13} & S_{14} & S_{15} & S_{16} \\ S_{21} & S_{22} & S_{23} & S_{24} & S_{25} & S_{26} \\ S_{31} & S_{32} & S_{33} & S_{34} & S_{35} & S_{36} \\ S_{41} & S_{42} & S_{43} & S_{44} & S_{45} & S_{46} \\ S_{51} & S_{52} & S_{53} & S_{54} & S_{55} & S_{56} \\ S_{61} & S_{62} & S_{63} & S_{64} & S_{65} & S_{66} \end{bmatrix} \begin{bmatrix} \boldsymbol{\sigma}_{xx} \\ \boldsymbol{\sigma}_{yy} \\ \boldsymbol{\sigma}_{zz} \\ \boldsymbol{\tau}_{xz} \\ \boldsymbol{\tau}_{xy} \end{bmatrix}$$

$$\sigma_i = C_{ij}\varepsilon_j, \quad i, j = 1...6$$

$$\begin{bmatrix} \sigma_1 \\ \sigma_2 \\ \sigma_3 \\ \sigma_4 \\ \sigma_5 \\ \sigma_6 \end{bmatrix} = \begin{bmatrix} C_{11} & C_{12} & C_{13} & C_{14} & C_{15} & C_{16} \\ C_{21} & C_{22} & C_{23} & C_{24} & C_{25} & C_{26} \\ C_{31} & C_{32} & C_{33} & C_{34} & C_{35} & C_{36} \\ C_{41} & C_{42} & C_{43} & C_{44} & C_{45} & C_{46} \\ C_{51} & C_{52} & C_{53} & C_{54} & C_{55} & C_{56} \\ C_{61} & C_{62} & C_{63} & C_{64} & C_{65} & C_{66} \end{bmatrix} \begin{bmatrix} \varepsilon_1 \\ \varepsilon_2 \\ \varepsilon_3 \\ \varepsilon_4 \\ \varepsilon_5 \\ \varepsilon_6 \end{bmatrix}$$

$$\sigma_i = C_{ij} \varepsilon_j, \quad i, j = 1...6$$

$\left[\sigma_{xx}\right]$		C_{11}	<i>C</i> ₁₂	<i>C</i> ₁₃	<i>C</i> ₁₄	<i>C</i> ₁₅	C_{16}	$\left[\boldsymbol{\varepsilon}_{xx} \right]$
σ_{yy}		<i>C</i> ₂₁	<i>C</i> ₂₂	<i>C</i> ₂₃	<i>C</i> ₂₄	<i>C</i> ₂₅	<i>C</i> ₂₆	$\boldsymbol{\varepsilon}_{yy}$
σ_{xy}	_	<i>C</i> ₃₁	<i>C</i> ₃₂	<i>C</i> ₃₃	<i>C</i> ₃₄	<i>C</i> ₃₅	<i>C</i> ₃₆	\mathcal{E}_{ZZ}
$ ight) au_{yz}$	_	C_{41}	C_{42}	<i>C</i> ₄₃	C_{44}	C_{45}	<i>C</i> ₄₆	γ_{yz}
$ au_{_{XZ}}$		C_{51}	C_{52}	<i>C</i> ₅₃	<i>C</i> ₅₄	<i>C</i> ₅₅	<i>C</i> ₅₆	γ_{xz}
$\left[\tau_{xy} \right]$		C_{61}	C_{62}	<i>C</i> ₆₃	C_{64}	<i>C</i> ₆₅	C_{66}	$\left[\gamma_{xy} \right]$

Principal Material Coordinate System



- Defined by the symmetry associated with fiber orientation
 - Usually labeled the 1-2-3 coordinate system
 - No "unusual" coupling in PMCS
 - Elastic & failure properties measured in the 1-2-3 coordinate system:

 $E_{11}, E_{22}, E_{33}, v_{12}, v_{13}, v_{23}, G_{12}, G_{13}, G_{23}$ $\sigma_{11}^{fT}, \sigma_{11}^{fC}, \sigma_{22}^{fT}, \sigma_{22}^{fC}, \sigma_{33}^{fT}, \sigma_{33}^{fC}, \tau_{12}^{f}, \tau_{13}^{f}, \tau_{23}^{f}$

Hooke's Law In Principal Material Coordinate System

• Since no "unusual" couplings, all of the following compliances equal zero in the principal material coordinate system:

$$S_{14} = S_{41}, S_{15} = S_{51}, S_{16} = S_{61}$$

$$S_{24} = S_{42}, S_{25} = S_{52}, S_{26} = S_{62}$$

$$S_{34} = S_{43}, S_{35} = S_{53}, S_{36} = S_{63}$$

$$S_{45} = S_{54}, S_{46} = S_{64}, S_{56} = S_{65}$$

Hooke's Law In Principal Material Coordinate System

• Since no "unusual" couplings, in the principal material coordinate system:

$$\begin{cases} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{23} \\ \gamma_{13} \\ \gamma_{12} \end{cases} = \begin{bmatrix} S_{11} & S_{12} & S_{13} & 0 & 0 & 0 \\ S_{12} & S_{22} & S_{23} & 0 & 0 & 0 \\ S_{13} & S_{23} & S_{33} & 0 & 0 & 0 \\ 0 & 0 & 0 & S_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & S_{55} & 0 \\ 0 & 0 & 0 & 0 & 0 & S_{66} \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \tau_{23} \\ \tau_{13} \\ \tau_{12} \end{bmatrix}$$

Hooke's Law In Principal Material Coordinate System

• Similarly:



Called "Orthotropic" if $E_{33} \neq E_{22}$:



Orthotropic Materials

$\left[\boldsymbol{\varepsilon}_{11} \right]$		<i>S</i> ₁₁	<i>S</i> ₁₂	<i>S</i> ₁₃	0	0	0	$\left[\sigma_{11}\right]$
ε_{22}		<i>S</i> ₁₂	<i>S</i> ₂₂	<i>S</i> ₂₃	0	0	0	σ_{22}
\mathcal{E}_{33}		<i>S</i> ₁₃	<i>S</i> ₂₃	<i>S</i> ₃₃	0	0	0	σ_{33}
γ_{23}		0	0	0	S_{44}	0	0	τ_{23}
γ_{13}		0	0	0	0	S ₅₅	0	τ_{13}
$\left[\gamma_{12}\right]$		0	0	0	0	0	S_{66}	$\left[\tau_{12} \right]$
(σ_{11})		$\begin{bmatrix} C_{11} \end{bmatrix}$	<i>C</i> ₁₂	<i>C</i> ₁₃	0	0	0	$\left \left(\varepsilon_{11}\right)\right $
$egin{pmatrix} \sigma_{11} \ \sigma_{22} \end{bmatrix}$		$\begin{bmatrix} C_{11} \\ C_{12} \end{bmatrix}$	C ₁₂ C ₂₂	C ₁₃ C ₂₃	0 0	0 0	0 - 0	$\begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \end{bmatrix}$
$egin{bmatrix} \sigma_{11} \ \sigma_{22} \ \sigma_{33} \end{bmatrix}$		$\begin{bmatrix} C_{11} \\ C_{12} \\ C_{13} \end{bmatrix}$	C ₁₂ C ₂₂ C ₂₃	C ₁₃ C ₂₃ C ₃₃	0 0 0	0 0 0	0 0 0	$\begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \end{bmatrix}$
$\left\{egin{array}{c} \sigma_{11} \ \sigma_{22} \ \sigma_{33} \ \tau_{23} \end{array} ight angle$	> =	$\begin{bmatrix} C_{11} \\ C_{12} \\ C_{13} \\ 0 \end{bmatrix}$	$C_{12} \\ C_{22} \\ C_{23} \\ 0$	$C_{13} \\ C_{23} \\ C_{33} \\ 0$	$0 \\ 0 \\ 0 \\ C_{44}$	0 0 0 0	0 0 0 0	$ \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{23} \end{bmatrix} $
$egin{pmatrix} \sigma_{11} \ \sigma_{22} \ \sigma_{33} \ \tau_{23} \ au_{13} \ \end{bmatrix}$	> =	$ \begin{bmatrix} C_{11} \\ C_{12} \\ C_{13} \\ 0 \\ 0 \end{bmatrix} $	$C_{12} \\ C_{22} \\ C_{23} \\ 0 \\ 0$	C_{13} C_{23} C_{33} 0 0	$egin{array}{c} 0 \\ 0 \\ 0 \\ C_{44} \\ 0 \end{array}$	$0 \\ 0 \\ 0 \\ 0 \\ C_{55}$	0 0 0 0 0	$ \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \gamma_{23} \\ \gamma_{13} \end{bmatrix} $

Called "Transversely Isotropic" if $E_{33} = E_{22}$



Transversely Isotropic Materials

$\left[\varepsilon_{11} \right]$		$\int S_{11}$	<i>S</i> ₁₂	<i>S</i> ₁₂	0	0	0	σ_{11}
ε_{22}		<i>S</i> ₁₂	<i>S</i> ₂₂	<i>S</i> ₂₃	0	0	0	σ_{22}
\mathcal{E}_{33}	L	<i>S</i> ₁₂	<i>S</i> ₂₃	<i>S</i> ₂₂	0	0	0	σ_{33}
γ_{23}	~ —	0	0	0	$2(S_{22}-S_{23})$	0	0	τ_{23}
γ_{13}		0	0	0	0	S ₆₆	0	τ_{13}
$\left(\gamma_{12}\right)$		0	0	0	0	0	S_{66} _	$\left(\tau_{12} \right)$

$\left[\sigma_{11}\right]$		C_{11}	<i>C</i> ₁₂	<i>C</i> ₁₂	0	0	0]	$\left(\mathcal{E}_{11} \right)$
σ_{22}		C_{12}	<i>C</i> ₂₂	<i>C</i> ₂₃	0	0	0	ε_{22}
σ_{33}	_	C_{12}	<i>C</i> ₂₃	<i>C</i> ₂₂	0	0	0	ε_{33}
τ_{23}	_	0	0	0	$(C_{22} - C_{23})/2$	0	0	γ_{23}
τ_{13}		0	0	0	0	<i>C</i> ₆₆	0	<i>Y</i> ₁₃
$\left(\tau_{12} \right)$		0	0	0	0	0	C_{66}	$\left[\gamma_{12}\right]$



• For thin, plate-like structures it is usually appropriate to assume all out-of-plane stresses are negligibly small (i.e., assume $\sigma_{33} = \tau_{13} = \tau_{23} = 0$)

• For plane stress conditions:





• Hooke's law can be "reduced" to:

$$\begin{cases} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{cases} = \begin{bmatrix} S_{11} & S_{12} & 0 \\ S_{12} & S_{22} & 0 \\ 0 & 0 & S_{66} \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \tau_{12} \end{bmatrix} \\ \varepsilon_{33} = S_{13}\sigma_{11} + S_{23}\sigma_{22} \end{cases}$$



$$[S] = \begin{bmatrix} S_{11} & S_{12} & 0 \\ S_{12} & S_{22} & 0 \\ 0 & 0 & S_{66} \end{bmatrix} = \text{the "reduced compliance matrix"}$$

• Inverting:

$$\begin{cases} \sigma_{11} \\ \sigma_{22} \\ \tau_{12} \end{cases} = \begin{bmatrix} Q_{11} & Q_{12} & 0 \\ Q_{12} & Q_{22} & 0 \\ 0 & 0 & Q_{66} \end{bmatrix} \begin{cases} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{cases}$$



$$[Q] = \begin{bmatrix} Q_{11} & Q_{12} & 0 \\ Q_{12} & Q_{22} & 0 \\ 0 & 0 & Q_{66} \end{bmatrix} = \text{the "reduced stiffness matrix"}$$

Strains Caused by Stress and/or Changes in Temperature and/or Moisture Content

• Three-dimensional stress-states (orthotropic mat'ls):



...Inverting

$$\begin{cases} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \tau_{23} \\ \tau_{13} \\ \tau_{12} \end{cases} = \begin{bmatrix} C_{11} & C_{12} & C_{13} & 0 & 0 & 0 \\ C_{12} & C_{22} & C_{23} & 0 & 0 & 0 \\ C_{13} & C_{23} & C_{33} & 0 & 0 & 0 \\ 0 & 0 & 0 & C_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & C_{55} & 0 \\ 0 & 0 & 0 & 0 & 0 & C_{66} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} - \Delta T \alpha_{11} - \Delta M \beta_{11} \\ \varepsilon_{22} - \Delta T \alpha_{22} - \Delta M \beta_{22} \\ \varepsilon_{33} - \Delta T \alpha_{33} - \Delta M \beta_{33} \\ \gamma_{23} \\ \gamma_{12} \end{bmatrix}$$

Strains Caused by Stress and/or Changes in Temperature and/or Moisture Content

• Reducing to plane stress conditions:

$$\begin{cases} \varepsilon_{11} \\ \varepsilon_{22} \\ \gamma_{12} \end{cases} = \begin{bmatrix} S_{11} & S_{12} & 0 \\ S_{12} & S_{22} & 0 \\ 0 & 0 & S_{66} \end{bmatrix} \begin{cases} \sigma_{11} \\ \sigma_{22} \\ \tau_{12} \end{cases} + \Delta T \begin{cases} \alpha_{11} \\ \alpha_{22} \\ 0 \end{cases} + \Delta M \begin{cases} \beta_{11} \\ \beta_{22} \\ 0 \end{cases} \end{cases}$$
$$\varepsilon_{33} = S_{13}\sigma_{11} + S_{23}\sigma_{22} + \Delta T\alpha_{33} + \Delta M\beta_{33}$$

Strains Caused by Stress and/or Changes in Temperature and/or Moisture Content

• Inverting:

$$\begin{cases} \sigma_{11} \\ \sigma_{22} \\ \tau_{12} \end{cases} = \begin{bmatrix} Q_{11} & Q_{12} & 0 \\ Q_{12} & Q_{22} & 0 \\ 0 & 0 & Q_{66} \end{bmatrix} \begin{cases} \varepsilon_{11} - \Delta T \alpha_{11} - \Delta M \beta_{11} \\ \varepsilon_{22} - \Delta T \alpha_{22} - \Delta M \beta_{22} \\ \gamma_{12} \end{cases}$$





$$\begin{cases} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \end{cases} = \begin{bmatrix} \overline{S}_{11} & \overline{S}_{12} & \overline{S}_{16} \\ \overline{S}_{12} & \overline{S}_{22} & \overline{S}_{26} \\ \overline{S}_{16} & \overline{S}_{26} & \overline{S}_{66} \end{bmatrix} \begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \tau_{12} \end{bmatrix} + \Delta T \begin{cases} \alpha_{xx} \\ \alpha_{yy} \\ \alpha_{xy} \end{cases} + \Delta M \begin{cases} \beta_{xx} \\ \beta_{yy} \\ \beta_{xy} \end{cases}$$
$$\varepsilon_{33} = \varepsilon_{zz} = S_{13}\sigma_{11} + S_{23}\sigma_{22} + \Delta T\alpha_{33} + \Delta M\beta_{33}$$
$$[\overline{S}_{ij}] = \text{Transformed, reduced compliance matrix}$$

$$\overline{S}_{11} = S_{11} \cos^4 \theta + (2S_{12} + S_{66}) \cos^2 \theta \sin^2 \theta + S_{22} \sin^4 \theta$$

$$\overline{S}_{12} = \overline{S}_{21} = S_{12} (\cos^4 \theta + \sin^4 \theta) + (S_{11} + S_{22} - S_{66}) \cos^2 \theta \sin^2 \theta$$

$$\overline{S}_{16} = \overline{S}_{61} = (2S_{11} - 2S_{12} - S_{66}) \cos^3 \theta \sin \theta - (2S_{22} - 2S_{12} - S_{66}) \cos \theta \sin^3 \theta$$

$$\overline{S}_{22} = S_{11} \sin^4 \theta + (2S_{12} + S_{66}) \cos^2 \theta \sin^2 \theta + S_{22} \cos^4 \theta$$

$$\overline{S}_{26} = \overline{S}_{62} = (2S_{11} - 2S_{12} - S_{66}) \cos \theta \sin^3 \theta - (2S_{22} - 2S_{12} - S_{66}) \cos^3 \theta \sin \theta$$

$$\overline{S}_{66} = 2(2S_{11} + 2S_{22} - 4S_{12} - S_{66}) \cos^2 \theta \sin^2 \theta + S_{66} (\cos^4 \theta + \sin^4 \theta)$$

$$\alpha_{xx} = \alpha_{11} \cos^2(\theta) + \alpha_{22} \sin^2(\theta)$$
$$\alpha_{yy} = \alpha_{11} \sin^2(\theta) + \alpha_{22} \cos^2(\theta)$$
$$\alpha_{xy} = 2\cos(\theta)\sin(\theta)(\alpha_{11} - \alpha_{22})$$

$$\beta_{xx} = \beta_{11} \cos^2(\theta) + \beta_{22} \sin^2(\theta)$$
$$\beta_{yy} = \beta_{11} \sin^2(\theta) + \beta_{22} \cos^2(\theta)$$
$$\beta_{xy} = 2\cos(\theta)\sin(\theta)(\beta_{11} - \beta_{22})$$

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...Inverting



$$\begin{cases} \sigma_{xx} \\ \sigma_{yy} \\ \tau_{xy} \end{cases} = \begin{bmatrix} \overline{Q}_{11} & \overline{Q}_{12} & \overline{Q}_{16} \\ \overline{Q}_{12} & \overline{Q}_{22} & \overline{Q}_{26} \\ \overline{Q}_{16} & \overline{Q}_{26} & \overline{Q}_{66} \end{bmatrix} \begin{cases} \varepsilon_{xx} - \Delta T \alpha_{xx} - \Delta M \beta_{xx} \\ \varepsilon_{yy} - \Delta T \alpha_{yy} - \Delta M \beta_{yy} \\ \gamma_{xy} - \Delta T \alpha_{xy} - \Delta M \beta_{xy} \end{cases}$$

 $\left[\overline{Q}_{ij}\right]$ = Transformed, reduced stiffness matrix

...Where

$$\begin{aligned} \overline{Q}_{11} &= Q_{11}\cos^4\theta + 2(Q_{12} + 2Q_{66})\cos^2\theta\sin^2\theta + Q_{22}\sin^4\theta \\ \overline{Q}_{12} &= \overline{Q}_{21} = Q_{12}(\cos^4\theta + \sin^4\theta) + (Q_{11} + Q_{22} - 4Q_{66})\cos^2\theta\sin^2\theta \\ \overline{Q}_{16} &= \overline{Q}_{61} = (Q_{11} - Q_{12} - 2Q_{66})\cos^3\theta\sin\theta - (Q_{22} - Q_{12} - 2Q_{66})\cos\theta\sin^3\theta \\ \overline{Q}_{22} &= Q_{11}\sin^4\theta + 2(Q_{12} + 2Q_{66})\cos^2\theta\sin^2\theta + Q_{22}\cos^4\theta \\ \overline{Q}_{26} &= \overline{Q}_{62} = (Q_{11} - Q_{12} - 2Q_{66})\cos\theta\sin^3\theta - (Q_{22} - Q_{12} - 2Q_{66})\cos^3\theta\sin\theta \\ \overline{Q}_{66} &= (Q_{11} + Q_{22} - 2Q_{12} - 2Q_{66})\cos^2\theta\sin^2\theta + Q_{66}(\cos^4\theta + \sin^4\theta) \end{aligned}$$

An Observation Regarding Stress Distributions assuming linear-elastic conditions with no body forces

• For *isotropic* materials/structures, stress distributions are independent of elastic material properties:

An Observation Regarding Stress Distributions assuming linear-elastic conditions with no body forces

- For <u>isotropic</u> materials/structures, stress distributions are independent of elastic material properties:
 - Can be seen from the theory of elasticity:

$$\sigma_{xx} = \frac{\partial^2 \phi}{\partial y^2}$$
 $\sigma_{yy} = \frac{\partial^2 \phi}{\partial x^2}$ $\tau_{xy} = -\frac{\partial^2 \phi}{\partial x \partial y}$

where:

 $\phi = \phi(x, y) =$ Airy stress function, which satisfies the biharmonic equation :

$$\frac{\partial^4 \phi}{\partial x^4} + 2 \frac{\partial^4 \phi}{\partial x^2 \partial y^2} + \frac{\partial^4 \phi}{\partial y^4} = 0$$

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(...unlike stresses, strains <u>are</u> a function of elastic material properties)

An Observation Regarding Stress Distributions assuming linear-elastic conditions with no body forces

 For <u>isotropic</u> materials/structures, stress distributions are independent of elastic properties:

An Observation Regarding Stress Distributions

assuming linear-elastic conditions with no body forces

For *isotropic* materials/structures, stress distributions are independent of elastic properties:

•The stress concentration factor for a circular hole in any large (i.e., "infinite") isotropic plate is always:



An Observation Regarding Stress Distributions assuming linear-elastic conditions with no body forces

- For *isotropic* materials/structures, stress distributions are independent of elastic properties:
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• In contrast, for anisotropic materials/structures, stress distributions are <u>not</u> independent of elastic material properties

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 - For example, the stress concentration factor for a circular hole in an anisotropic plate varies widely...using typical properties for graphite-epoxy:

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 - For example, the stress concentration factor for a circular hole in an anisotropic plate varies widely...using typical properties for graphite-epoxy:

•SCF for circular hole in large [0°] panel: ~ 6-8



An Observation Regarding Stress Distributions assuming linear-elastic conditions with no body forces

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 - For example, the stress concentration factor for a circular hole in an anisotropic plate varies widely...using typical properties for graphite-epoxy:

•SCF for circular hole in large [0°] panel: ~ 6-8 •SCF for circular hole in large [90°] panel: ~ 1.5 - 2.5

Macroscopic Failure Theories

 Dozens of failure theories have been proposed...none are universally accepted

•Hinton, M.J., Kaddour, A.S., and Soden, P.D., *Failure Criteria in Fibre Reinforced Polymer Composites: The World-Wide failure Exercise*, Elsevier, ISBN 0-08-044475-X (2004)

•Three common failure criterion described in textbook:

- •Maximum stress failure criterion
- •Tsai-Hill failure criterion
- •Tsai-Wu failure criterion

Macroscopic Failure Theories Maximum Stress Failure Criterion (plane stress form)

• Failure does not occur if:

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Macroscopic Failure Theories Tsai-Hill Failure Criterion (plane stress form)

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• Failure does not occur if:

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$$X_{1}\sigma_{11} + X_{2}\sigma_{22} + X_{11}\sigma_{11}^{2} + X_{22}\sigma_{22}^{2} + X_{66}\tau_{12}^{2} + 2X_{12}\sigma_{11}\sigma_{22} < 1$$



Macroscopic Failure Theories: Comparisons Off-axis Gr/Ep specimen subject to uniaxial stress



Macroscopic Failure Theories: Comparisons Off-axis Gr/Ep specimen subjected to pure shear stress



Multi-ply, Multi-directional Composite Laminates



Defining Ply Interface Positions



Describing Stacking Sequences



Describing Stacking Sequences





Kirchhoff Hypothesis

"a straight line which is initially perpendicular to the midplane of a thin plate remains straight and perpendicular to the midplane after deformation"

• Ultimately allows us to calculate the strain at any through-thickness position z:

$$\begin{cases} \boldsymbol{\varepsilon}_{xx} \\ \boldsymbol{\varepsilon}_{yy} \\ \boldsymbol{\gamma}_{xy} \end{cases} = \begin{cases} \boldsymbol{\varepsilon}_{xx}^{o} \\ \boldsymbol{\varepsilon}_{yy}^{o} \\ \boldsymbol{\gamma}_{xy}^{o} \end{cases} + z \begin{cases} \boldsymbol{\kappa}_{xx} \\ \boldsymbol{\kappa}_{yy} \\ \boldsymbol{\kappa}_{xy} \end{cases}$$

where :

 $\varepsilon_{xx}^{o}, \varepsilon_{yy}^{o}, \gamma_{xy}^{o} = \text{midplane strains}$ $\kappa_{xx}, \kappa_{yy}, \kappa_{xy} = \text{midplane curvatures}$



Ply Strains

• Strains at ply interfaces are usually of greatest interest:



Ply Stresses

• Ply stresses can be calculated using Hooke's law:



Ply Stresses

• Ply stresses can be calculated using Hooke's law:





- Loads considered are restricted to those that lead to plane stress conditions...two types:
 - Stress resultants, N_{xx} , N_{yy} , and N_{xy}units = force/length





- Loads considered are restricted to those that lead to plane stress conditions...two types:
 - Moment resultants, M_{xx} , M_{yy} , and M_{xy}units = force-length/length



• It can be shown:

$$N_{xx} = \int_{-t/2}^{t/2} \sigma_{xx} \, dz$$

• Hooke's law:

$$\begin{cases} \sigma_{xx} \\ \sigma_{yy} \\ \tau_{xy} \end{cases} = \begin{bmatrix} \overline{\underline{Q}}_{11} & \overline{\underline{Q}}_{12} & \overline{\underline{Q}}_{16} \\ \overline{\underline{Q}}_{12} & \overline{\underline{Q}}_{22} & \overline{\underline{Q}}_{26} \\ \overline{\underline{Q}}_{16} & \overline{\underline{Q}}_{26} & \overline{\underline{Q}}_{66} \end{bmatrix} \begin{cases} \varepsilon_{xx} - \Delta T \alpha_{xx} - \Delta M \beta_{xx} \\ \varepsilon_{yy} - \Delta T \alpha_{yy} - \Delta M \beta_{yy} \\ \gamma_{xy} - \Delta T \alpha_{xy} - \Delta M \beta_{xy} \end{cases}$$

• Substituting for σ_{xx} and integrating in piece-wise fashion:

$$N_{xx} = A_{11}\varepsilon_{xx}^{o} + A_{12}\varepsilon_{yy}^{o} + A_{16}\gamma_{xy}^{o} + B_{11}\kappa_{xx} + B_{12}\kappa_{yy} + B_{16}\kappa_{xy} - N_{xx}^{T} - N_{xx}^{M}$$

• Where:

$$A_{11} = \{ \overline{Q}_{11} \}_{1} [z_{1} - z_{0}] + (\overline{Q}_{11})_{2} [z_{2} - z_{1}] + (\overline{Q}_{11})_{3} [z_{3} - z_{2}] + \dots + (\overline{Q}_{11})_{n} [z_{n} - z_{n-1}] \}$$

$$A_{12} = \{ \overline{Q}_{12} \}_{1} [z_{1} - z_{0}] + (\overline{Q}_{12})_{2} [z_{2} - z_{1}] + (\overline{Q}_{12})_{3} [z_{3} - z_{2}] + \dots + (\overline{Q}_{12})_{n} [z_{n} - z_{n-1}] \}$$

$$A_{16} = \{ \overline{Q}_{16} \}_{1} [z_{1} - z_{0}] + (\overline{Q}_{16})_{2} [z_{2} - z_{1}] + (\overline{Q}_{16})_{3} [z_{3} - z_{2}] + \dots + (\overline{Q}_{16})_{n} [z_{n} - z_{n-1}] \}$$

$$B_{11} = \frac{1}{2} \left\{ \overline{Q}_{11} \right\}_{1} [z_{1}^{2} - z_{0}^{2}] + \left(\overline{Q}_{11} \right)_{2} [z_{2}^{2} - z_{1}^{2}] + \left(\overline{Q}_{11} \right)_{3} [z_{3}^{2} - z_{2}^{2}] + \dots + \left(\overline{Q}_{11} \right)_{n} [z_{n}^{2} - z_{n-1}^{2}] \right\}$$

$$B_{12} = \frac{1}{2} \left\{ \overline{Q}_{12} \right\}_{1} [z_{1}^{2} - z_{0}^{2}] + \left(\overline{Q}_{12} \right)_{2} [z_{2}^{2} - z_{1}^{2}] + \left(\overline{Q}_{12} \right)_{3} [z_{3}^{2} - z_{2}^{2}] + \dots + \left(\overline{Q}_{12} \right)_{n} [z_{n}^{2} - z_{n-1}^{2}] \right\}$$

$$B_{16} = \frac{1}{2} \left\{ \overline{Q}_{16} \right\}_{1} [z_{1}^{2} - z_{0}^{2}] + \left(\overline{Q}_{16} \right)_{2} [z_{2}^{2} - z_{1}^{2}] + \left(\overline{Q}_{16} \right)_{3} [z_{3}^{2} - z_{2}^{2}] + \dots + \left(\overline{Q}_{16} \right)_{n} [z_{n}^{2} - z_{n-1}^{2}] \right\}$$

$$N_{xx}^{T} \equiv \Delta T \sum_{k=1}^{n} \left\{ \overline{Q}_{11} \alpha_{xx} + \overline{Q}_{12} \alpha_{yy} + \overline{Q}_{16} \alpha_{xy} \right\}_{k} \left[z_{k} - z_{k-1} \right] \right\}$$
$$N_{xx}^{M} \equiv \Delta M \sum_{k=1}^{n} \left\{ \overline{Q}_{11} \beta_{xx} + \overline{Q}_{12} \beta_{yy} + \overline{Q}_{16} \beta_{xy} \right\}_{k} \left[z_{k} - z_{k-1} \right] \right\}$$

• It can be shown:

$$M_{xx} = \int_{-t/2}^{t/2} \sigma_{xx} z dz$$

• Hooke's law:

$$\begin{cases} \sigma_{xx} \\ \sigma_{yy} \\ \tau_{xy} \end{cases} = \begin{bmatrix} \overline{\underline{Q}}_{11} & \overline{\underline{Q}}_{12} & \overline{\underline{Q}}_{16} \\ \overline{\underline{Q}}_{12} & \overline{\underline{Q}}_{22} & \overline{\underline{Q}}_{26} \\ \overline{\underline{Q}}_{16} & \overline{\underline{Q}}_{26} & \overline{\underline{Q}}_{66} \end{bmatrix} \begin{cases} \varepsilon_{xx} - \Delta T \alpha_{xx} - \Delta M \beta_{xx} \\ \varepsilon_{yy} - \Delta T \alpha_{yy} - \Delta M \beta_{yy} \\ \gamma_{xy} - \Delta T \alpha_{xy} - \Delta M \beta_{xy} \end{cases}$$

• Substituting for σ_{xx} and integrating in piece-wise fashion:

 $M_{xx} = B_{11}\varepsilon_{xx}^{o} + B_{12}\varepsilon_{yy}^{o} + B_{16}\gamma_{xy}^{o} + D_{11}\kappa_{xx} + D_{12}\kappa_{yy} + D_{16}\kappa_{xy} - M_{xx}^{T} - M_{xx}^{M}$

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Laminate Loading

• Where:

$$B_{11} = \frac{1}{2} \left\{ \overline{Q}_{11} \right\}_{1} [z_{1}^{2} - z_{0}^{2}] + \left(\overline{Q}_{11} \right)_{2} [z_{2}^{2} - z_{1}^{2}] + \left(\overline{Q}_{11} \right)_{3} [z_{3}^{2} - z_{2}^{2}] + \dots + \left(\overline{Q}_{11} \right)_{n} [z_{n}^{2} - z_{n-1}^{2}] \right\}$$

$$B_{12} = \frac{1}{2} \left\{ \overline{Q}_{12} \right\}_{1} [z_{1}^{2} - z_{0}^{2}] + \left(\overline{Q}_{12} \right)_{2} [z_{2}^{2} - z_{1}^{2}] + \left(\overline{Q}_{12} \right)_{3} [z_{3}^{2} - z_{2}^{2}] + \dots + \left(\overline{Q}_{12} \right)_{n} [z_{n}^{2} - z_{n-1}^{2}] \right\}$$

$$B_{16} = \frac{1}{2} \left\{ \overline{Q}_{16} \right\}_{1} [z_{1}^{2} - z_{0}^{2}] + \left(\overline{Q}_{16} \right)_{2} [z_{2}^{2} - z_{1}^{2}] + \left(\overline{Q}_{16} \right)_{3} [z_{3}^{2} - z_{2}^{2}] + \dots + \left(\overline{Q}_{16} \right)_{n} [z_{n}^{2} - z_{n-1}^{2}] \right\}$$

$$D_{11} = \frac{1}{3} \left\{ \overline{Q}_{11} \right\}_{1} [z_{1}^{3} - z_{0}^{3}] + (\overline{Q}_{11})_{2} [z_{2}^{3} - z_{1}^{3}] + (\overline{Q}_{11})_{3} [z_{3}^{3} - z_{2}^{3}] + \dots + (\overline{Q}_{11})_{n} [z_{n}^{3} - z_{n-1}^{3}] \right\}$$

$$D_{12} = \frac{1}{3} \left\{ \overline{Q}_{12} \right\}_{1} [z_{1}^{3} - z_{0}^{3}] + (\overline{Q}_{12})_{2} [z_{2}^{3} - z_{1}^{3}] + (\overline{Q}_{12})_{3} [z_{3}^{3} - z_{2}^{3}] + \dots + (\overline{Q}_{12})_{n} [z_{n}^{3} - z_{n-1}^{3}] \right\}$$

$$D_{16} = \frac{1}{3} \left\{ \overline{Q}_{16} \right\}_{1} [z_{1}^{3} - z_{0}^{3}] + (\overline{Q}_{16})_{2} [z_{2}^{3} - z_{1}^{3}] + (\overline{Q}_{16})_{3} [z_{3}^{3} - z_{2}^{3}] + \dots + (\overline{Q}_{16})_{n} [z_{n}^{3} - z_{n-1}^{3}] \right\}$$

$$M_{xx}^{T} = \frac{\Delta T}{2} \sum_{k=1}^{n} \left\{ \overline{Q}_{11} \alpha_{xx} + \overline{Q}_{12} \alpha_{yy} + \overline{Q}_{16} \alpha_{xy} \right\}_{k} \left[z_{k}^{2} - z_{k-1}^{2} \right] \right\}$$
$$M_{xx}^{M} = \frac{\Delta M}{2} \sum_{k=1}^{n} \left\{ \overline{Q}_{11} \beta_{xx} + \overline{Q}_{12} \beta_{yy} + \overline{Q}_{16} \beta_{xy} \right\}_{k} \left[z_{k}^{2} - z_{k-1}^{2} \right] \right\}$$

• Process repeated for all stress and moment resultants, finally resulting in:

$\left(N_{xx}\right)$	}=	A_{11}	A_{12}	A_{16}	<i>B</i> ₁₁	<i>B</i> ₁₂	B_{16}	$\left[\varepsilon_{xx}^{o} \right]$	$\left[\begin{array}{c} N_{xx}^T \\ \end{array} \right]$	N_{xx}^M	
N _{yy}		<i>A</i> ₁₂	A ₂₂	A_{26}	<i>B</i> ₁₂	<i>B</i> ₂₂	<i>B</i> ₂₆	$\left \boldsymbol{\varepsilon}_{yy}^{o} \right $	N_{yy}^T	N_{yy}^M	
$ N_{xy} $		A_{16}	A_{26}	A ₆₆	<i>B</i> ₁₆	<i>B</i> ₂₆	<i>B</i> ₆₆	$\left \gamma^{o}_{xy} \right _{-}$	$\int N_{xy}^T$	N_{xy}^M	
M_{xx}		<i>B</i> ₁₁	<i>B</i> ₁₂	<i>B</i> ₁₆	<i>D</i> ₁₁	<i>D</i> ₁₂	<i>D</i> ₁₆	\mathcal{K}_{XX}	M_{xx}^{T}	M_{xx}^{M}	ſ
M_{yy}		<i>B</i> ₁₂	<i>B</i> ₂₂	<i>B</i> ₂₆	<i>D</i> ₁₂	<i>D</i> ₂₂	<i>D</i> ₂₆	Kyy	M_{yy}^{T}	$M \frac{M}{yy}$	
$\left[M_{xy}\right]$		<i>B</i> ₁₆	<i>B</i> ₂₆	<i>B</i> ₆₆	<i>D</i> ₁₆	<i>D</i> ₂₆	D_{66}	$\left[\kappa_{xy} \right]$	M_{xy}^{T}	M_{xy}^{M}	

• Inverting:

$$\begin{cases} \boldsymbol{\varepsilon}_{xx}^{o} \\ \boldsymbol{\varepsilon}_{yy}^{o} \\ \boldsymbol{\gamma}_{xy}^{o} \\ \boldsymbol{\kappa}_{xx} \\ \boldsymbol{\kappa}_{yy} \\ \boldsymbol{\kappa}_{xy} \end{cases} = \begin{bmatrix} a_{11} & a_{12} & a_{16} & b_{11} & b_{12} & b_{16} \\ a_{12} & a_{22} & a_{26} & b_{21} & b_{22} & b_{26} \\ a_{16} & a_{26} & a_{66} & b_{61} & b_{62} & b_{66} \\ b_{11} & b_{21} & b_{61} & d_{11} & d_{12} & d_{16} \\ b_{12} & b_{22} & b_{62} & d_{12} & d_{22} & d_{26} \\ b_{16} & b_{26} & b_{66} & d_{16} & d_{26} & d_{66} \end{bmatrix} \begin{bmatrix} N_{xx} + N_{xx}^{T} + N_{xx}^{M} \\ N_{yy} + N_{yy}^{T} + N_{yy}^{M} \\ N_{xy} + N_{xy}^{T} + N_{xy}^{M} \\ M_{xx} + M_{xx}^{T} + M_{xx}^{M} \\ M_{yy} + M_{yy}^{T} + M_{yy}^{M} \\ M_{xy} + M_{xy}^{T} + M_{xy}^{M} \\ \end{bmatrix}$$

Simplifications Due to Stacking Sequence

- Various terms within the [*ABD*] and [*abd*] matrices are always zero for certain stacking sequences (see Section 6.7)
- The most important simplification occurs for *symmetric* laminates...in this case:

$$\begin{bmatrix} N_{xx} \\ N_{yy} \\ N_{xy} \\ M_{xx} \\ M_{yy} \\ M_{xy} \end{bmatrix} = \begin{bmatrix} A_{11} & A_{12} & A_{16} & 0 & 0 & 0 \\ A_{12} & A_{22} & A_{26} & 0 & 0 & 0 \\ A_{16} & A_{26} & A_{66} & 0 & 0 & 0 \\ 0 & 0 & 0 & D_{11} & D_{12} & D_{16} \\ 0 & 0 & 0 & D_{12} & D_{22} & D_{26} \\ 0 & 0 & 0 & D_{16} & D_{26} & D_{66} \end{bmatrix} \begin{bmatrix} \varepsilon_{xx}^{o} \\ \varepsilon_{yy}^{o} \\ \varepsilon_{yy}^{o} \\ \varepsilon_{yy}^{o} \\ \varepsilon_{xy}^{o} \\ \kappa_{xy} \\ \kappa_{xy} \\ \kappa_{xy} \end{bmatrix} = \begin{bmatrix} N_{xx}^{T} \\ N_{xy}^{M} \\ N_{xy}^{M} \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

Simplifications Due to Stacking Sequence

- Various terms within the [*ABD*] and [*abd*] matrices are always zero for certain stacking sequences (see Section 6.7)
- The most important simplification occurs for symmetric laminates...in this case:

$$\begin{bmatrix} \boldsymbol{\varepsilon}_{xx}^{o} \\ \boldsymbol{\varepsilon}_{yy}^{o} \\ \boldsymbol{\gamma}_{xy}^{o} \\ \boldsymbol{\kappa}_{xx} \\ \boldsymbol{\kappa}_{yy} \\ \boldsymbol{\kappa}_{xy} \end{bmatrix} = \begin{bmatrix} a_{11} & a_{12} & a_{16} & 0 & 0 & 0 \\ a_{12} & a_{22} & a_{26} & 0 & 0 & 0 \\ a_{16} & a_{26} & a_{66} & 0 & 0 & 0 \\ 0 & 0 & 0 & d_{11} & d_{12} & d_{16} \\ 0 & 0 & 0 & d_{12} & d_{22} & d_{26} \\ 0 & 0 & 0 & d_{16} & d_{26} & d_{66} \end{bmatrix} \begin{bmatrix} N_{xx} + N_{xx}^{T} + N_{xx}^{M} \\ N_{yy} + N_{yy}^{T} + N_{yy}^{M} \\ N_{xy} + N_{xy}^{T} + N_{xy}^{M} \\ M_{xx} \\ M_{yy} \\ M_{xy} \end{bmatrix}$$

Effective Laminate Properties

- Effective elastic properties of a laminate can be determined using elements of the $[a_{ij}]$ an $[d_{ij}]$ matrices
- Extensional (in-plane):

$$\overline{E}_{xx}^{ex} = \frac{1}{ta_{11}} \qquad \overline{v}_{xy}^{ex} = \frac{-a_{12}}{a_{11}} \qquad \overline{\eta}_{xx,xy}^{ex} = \frac{a_{16}}{a_{11}}$$
$$\overline{E}_{yy}^{ex} = \frac{1}{ta_{22}} \qquad \overline{v}_{yx}^{ex} = \frac{-a_{12}}{a_{22}} \qquad \overline{\eta}_{yy,xy}^{ex} = \frac{a_{26}}{a_{22}}$$
$$\overline{G}_{xy} = \frac{1}{ta_{66}} \qquad \overline{\eta}_{xy,xx}^{ex} = \frac{a_{16}}{a_{66}} \qquad \overline{\eta}_{xy,yy}^{ex} = \frac{a_{26}}{a_{66}}$$

Effective Laminate Properties

- Effective elastic properties of a laminate can be determined using elements of the $[a_{ij}]$ an $[d_{ij}]$ matrices
- Flexural (bending):

$$\overline{E}_{xx}^{fl} = \frac{12}{t^3 d_{11}} \qquad \overline{\nu}_{xy}^{fl} = \frac{-d_{12}}{d_{11}} \qquad \overline{\eta}_{xx,xy}^{fl} = \frac{d_{16}}{d_{11}}$$
$$\overline{E}_{yy}^{fl} = \frac{12}{t^3 d_{22}} \qquad \overline{\nu}_{yx}^{fl} = \frac{-d_{12}}{d_{22}} \qquad \overline{\eta}_{yy,xy}^{fl} = \frac{d_{26}}{d_{22}}$$



Program CLT

• Most of the topics included in this review are implemented in the program CLT (<u>Classical Lamination Theory</u>)

Laminate Damage Progression and Failure



3-D Stress-State Exists Near a Free-edge and Complicate Failure Predictions (see Section 6.13)



3-D Stress-State Exists Near a Free-edge and Complicate Failure Predictions *(see Section 6.13)*

(Typical results for a [45/-45]_s laminate subject to uniaxial loading)



First-ply Failure Loads and/or First-ply Failure Envelopes can be Predicted Using Program LAMFAIL


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Environmental Effects Dramatically Effect Predicted First-ply Failure Loads



Last-Ply Failure Loads Can be Predicted Using the Ply-Discount Scheme (Program LAMFAIL)



Predicted stress-strain curve for a $[0/30/60]_{s}$ laminate